



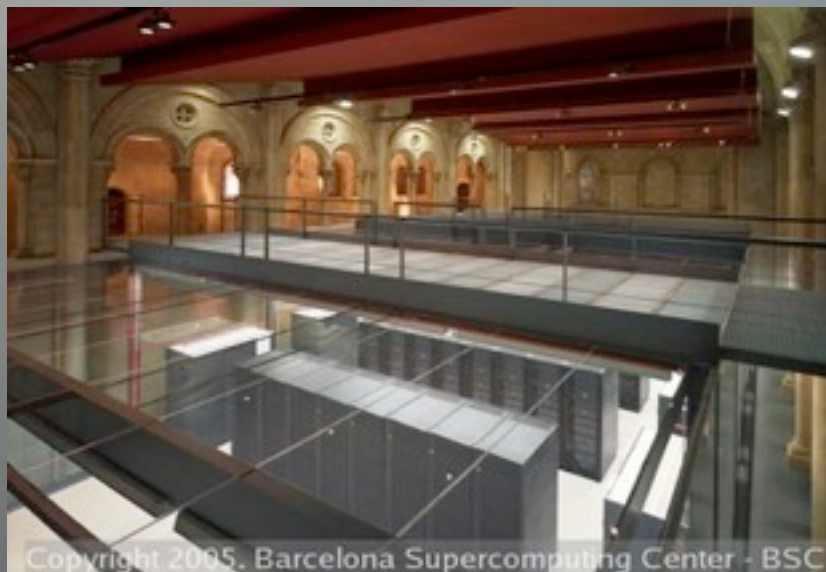
# High Performance Computational Fluid Dynamics at BSC –Driving & Sailing

CASE Department  
Barcelona Supercomputing Center – Centro  
Nacional de Supercomputación (BSC–CNS)  
Spain

Contact:

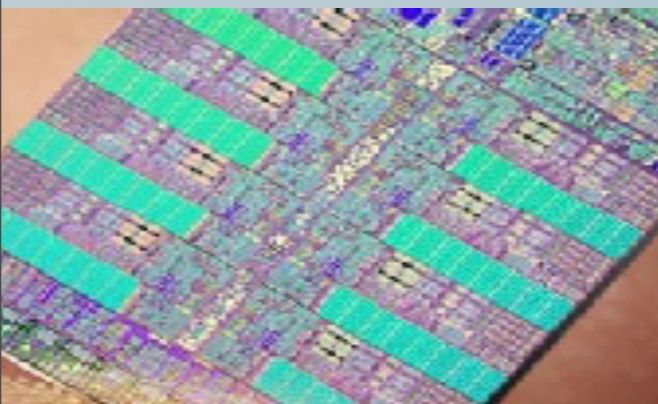
[mariano.vazquez@bsc.es](mailto:mariano.vazquez@bsc.es)

[herbert.owen@bsc.es](mailto:herbert.owen@bsc.es)

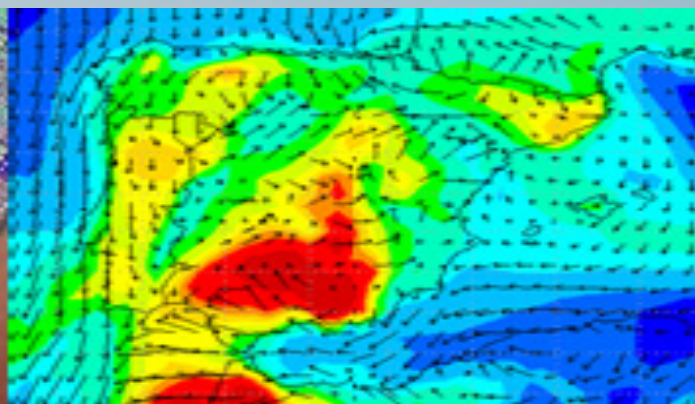




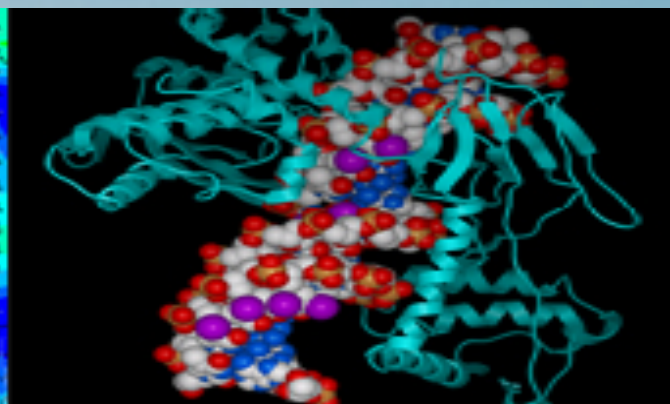
## BSC-CNS departments



**COMPUTER SCIENCES**



**EARTH SCIENCES**



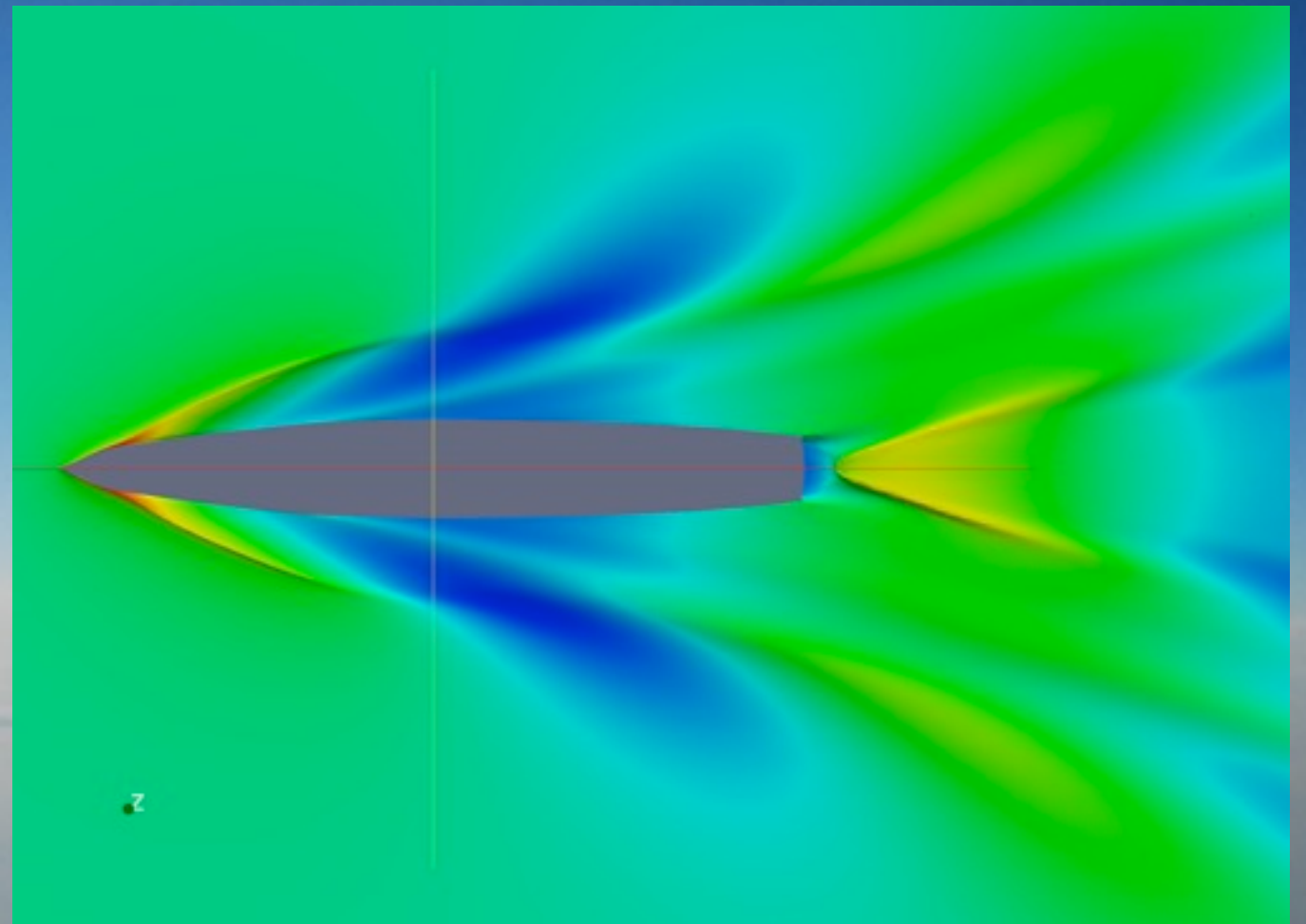
**LIFE SCIENCES**



**COMPUTER APPLICATIONS**

Computational Mechanics  
Applied Computer Science  
Optimization

## Computer Applications in Science and Engineering (CASE) Department



Interdisciplinary research unit of the BSC–CNS

Our mission:

To develop **computational tools** to simulate **highly complex** problems  
seamlessly adapted to run onto **high-end parallel supercomputers**



## Research lines

Physical and Numerical Modeling

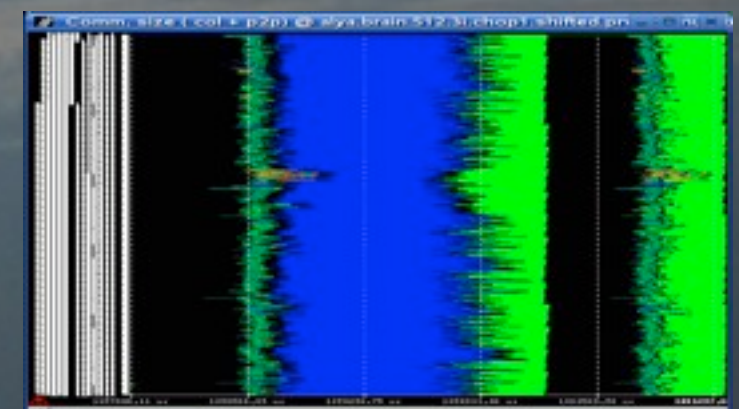
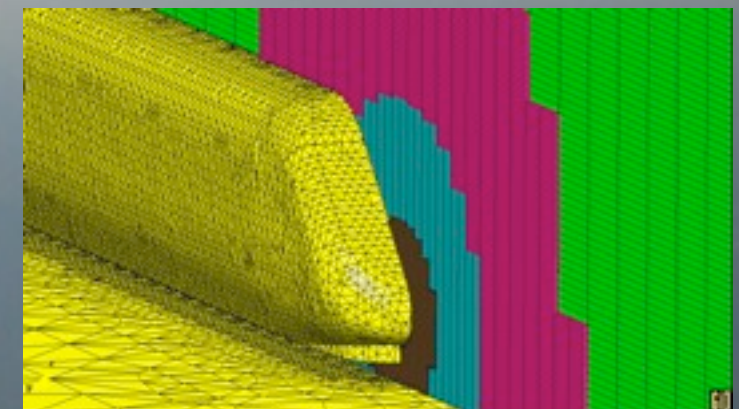
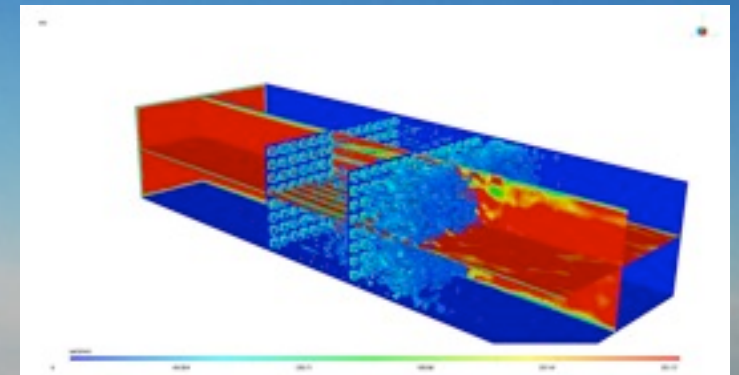
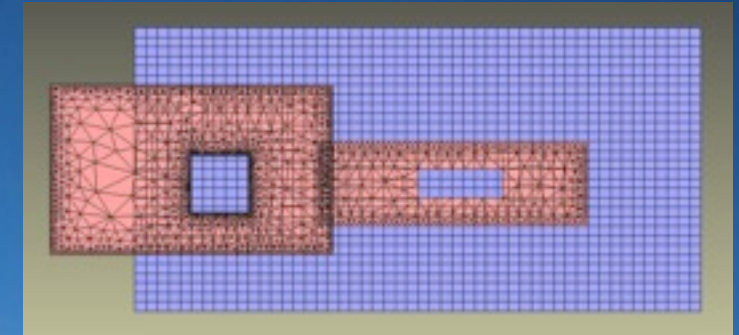
Mesh Generation and Numerical Solution Algorithms

High Performance Computing in Computational Mechanics

Parallelization in Distributed and Shared memory machines

Optimization

Algebraic Solvers





## Application lines

Bio Mechanics

Meteorology

Building, Energy & Environment

Aerospace Industry

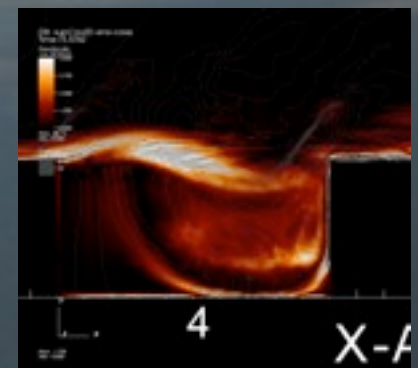
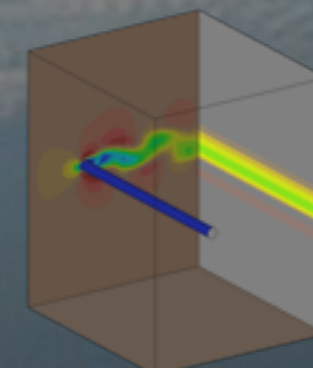
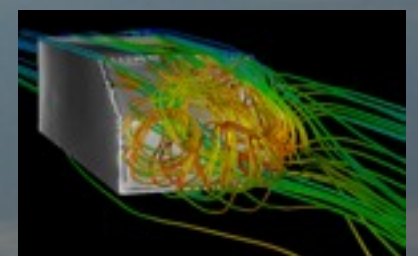
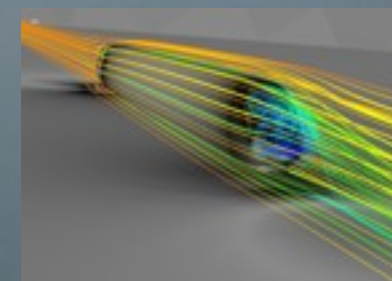
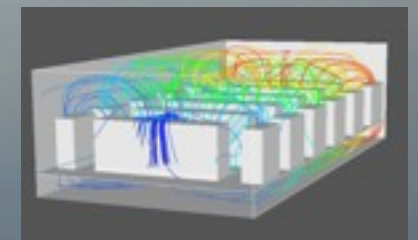
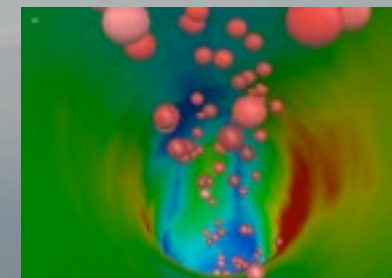
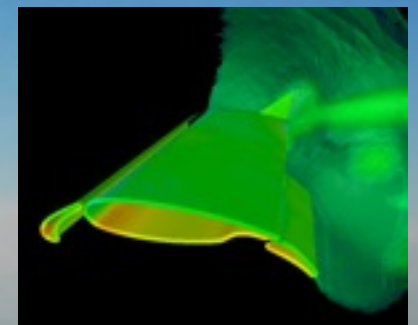
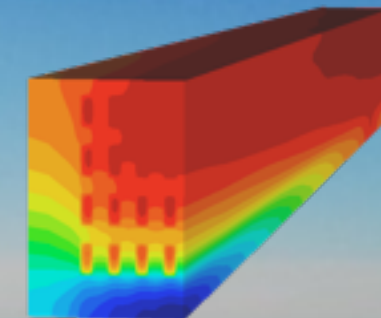
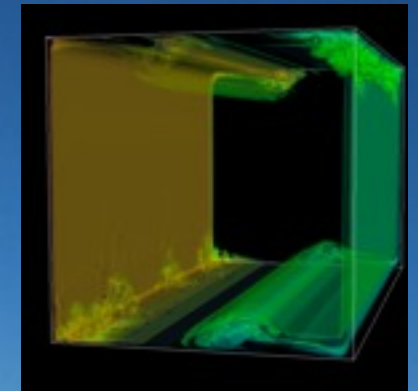
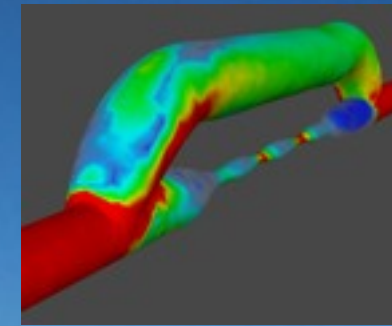
Land Vehicles Industry

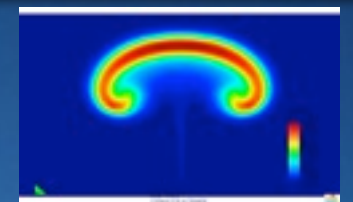
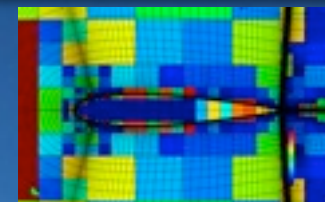
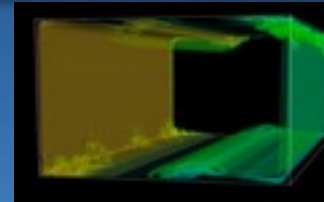
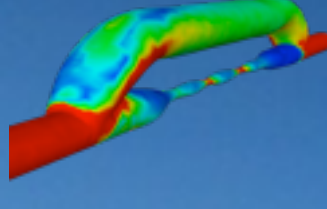
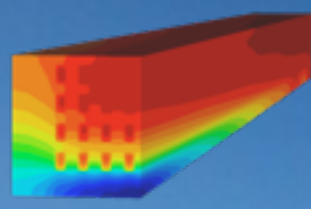
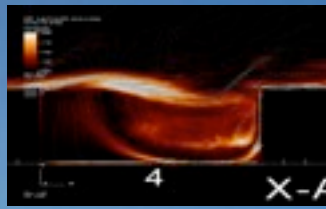
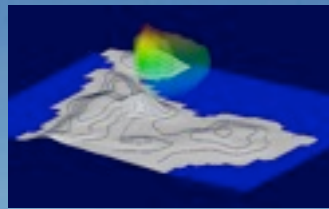
Ship Hydrodynamics

Oil Industry

Material Science

Artificial Societies





BSC In-house code

Multi-physics modular code for High Performance Computational Mechanics & Design

Numerical solution of PDE's

Variational methods are preferred (FEM)

Hybrid meshes, non-conforming meshes

Explicit and Implicit formulations

Coupling between multi-physics (loose or strong)

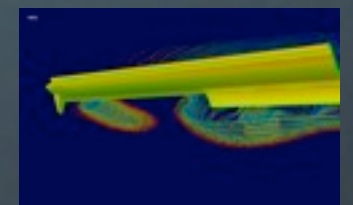
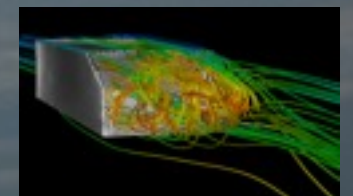
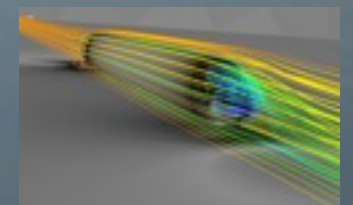
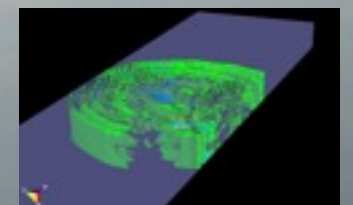
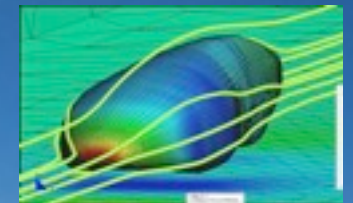
Advanced meshing issues

Parallelization by MPI and OpenMP

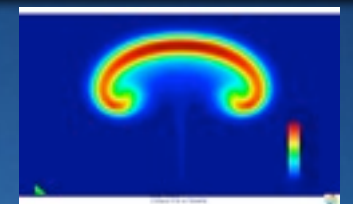
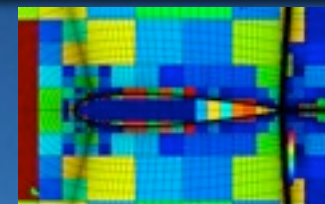
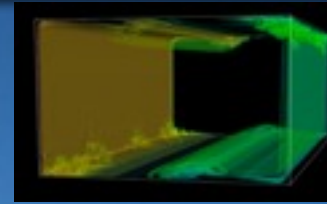
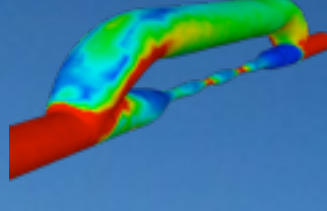
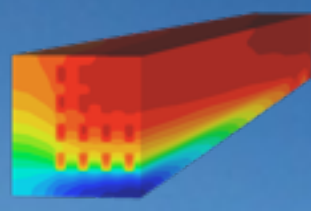
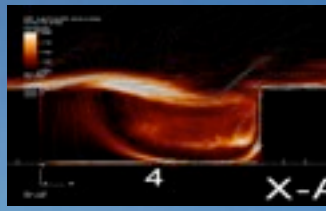
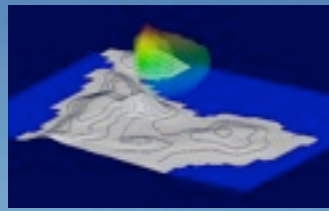
Automatic mesh partition using Metis

Portability is a must

Porting to new architectures: CELL, GPUs, ...







Compressible / Incompressible Navier–Stokes

Turbulence (RANS & LES models)

Free surface

Non-linear solid mechanics

Wave equation

Heat transport

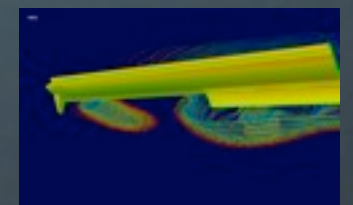
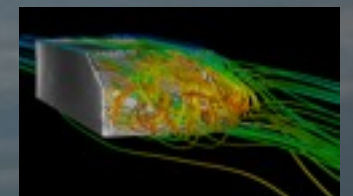
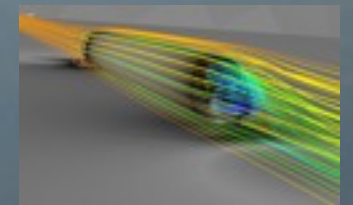
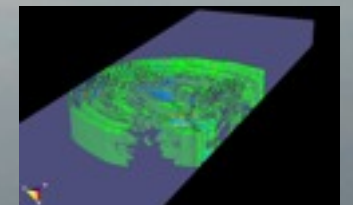
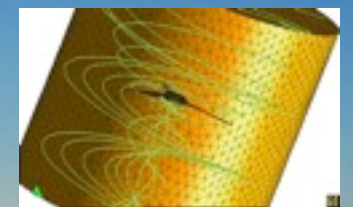
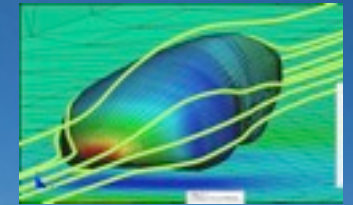
Excitable media

Helmholz equation

Particle transport

Chemical reactions

Many bodies



## **RANS: Reynolds Average Navier Stokes**

Obtains a stationary solution that is representative of the mean flow.

Models the effect of the fluctuating flow on the mean flow.

Very limited application to transient simulations.

Much cheaper computationally. At least 1 order of magnitude.

Wide spread use in the industry. Available in all commercial codes.

More robust. Easier to use.

Poor results when there are important separated regions.

Very limited application to transient simulations such as Aeroacoustics.

Some models: Spalart Allmaras ; SST k- $\omega$  ; k- $\epsilon$



## **LES: Large Eddy Simulation**

Transient simulations: at least 2nd order time accurate Methods.

Also higher accuracy (less diffusive numerical schemes) in space.

The evolution of the Large Eddies is simulated.

Must be averaged over large time periods of time.

Much higher computational cost. Highly scalable and efficient codes required.

Can not be used to model the flow close to the wall for High Reynolds problems due to computational cost. (At least for the next 2 decades).

Wall Laws for LES.

Hybrid RANS / LES. Detached Eddy Simulation – DES

Some models: Smagorinsky ; WALE ; Sigma ; VMS (MILES)

## Discretization and Numerical method

- VMS – ASGS – Stabilized finite element formulation.
- Unstructured Tetrahedras, Prisms and Pyramids.

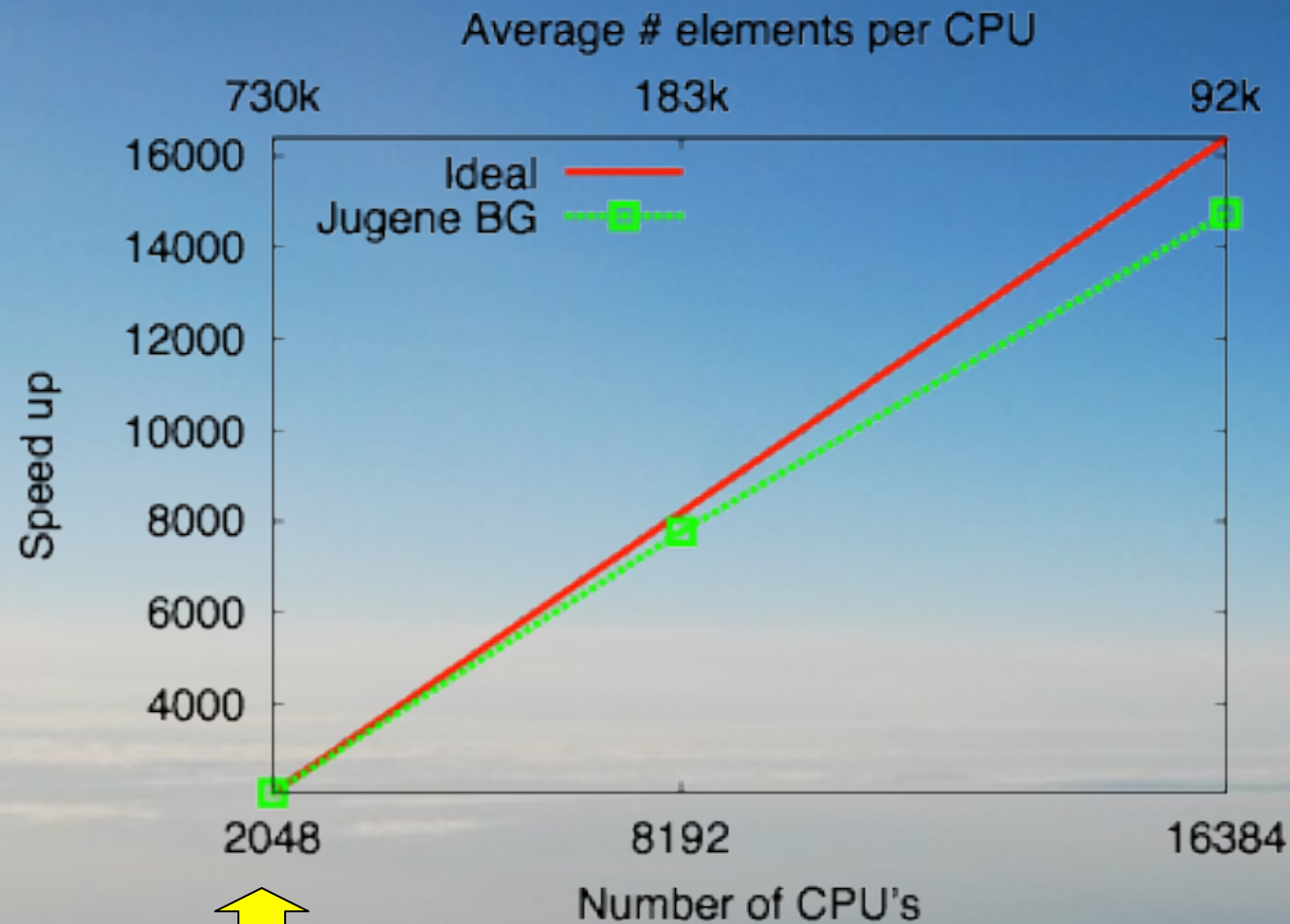
## Flow Algorithm

- Ortomin(1) solver for the Pressure Schur complement.

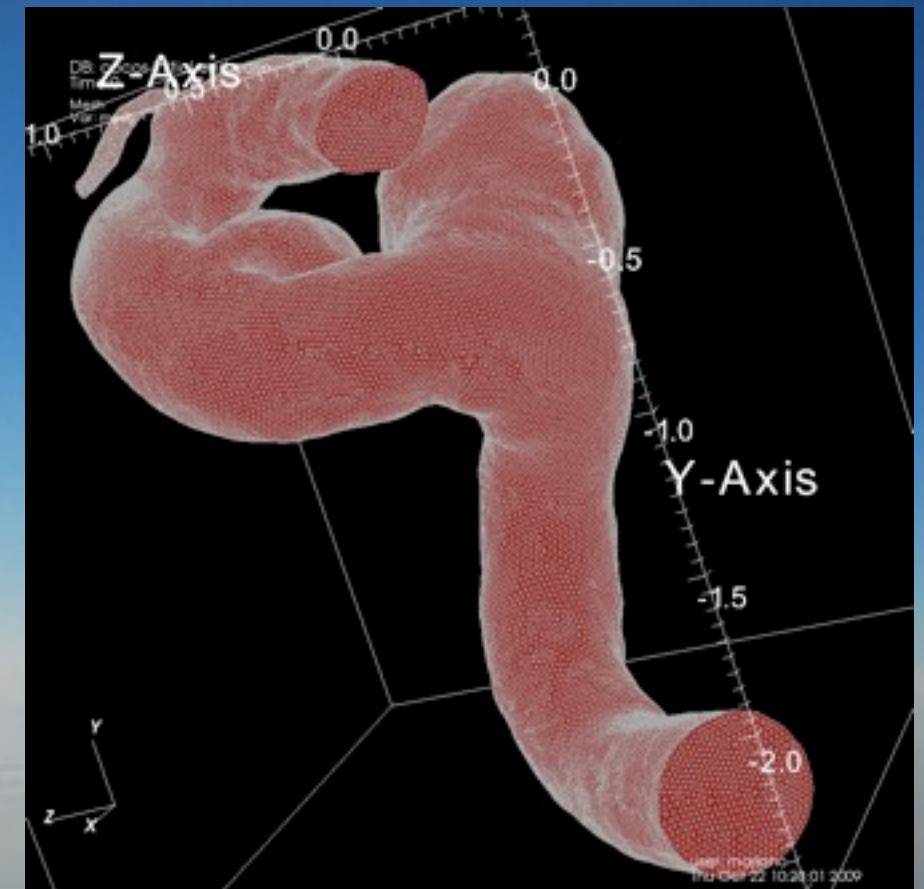
It is an extension of fractional step or SIMPLE techniques for incompressible flows where the relaxation parameter is obtained dynamically from the Orthomin algorithm.

- GMRES for velocity + Deflated CG for pressure.





1 hour to less  
than 2 seconds  
with no added  
cost!!!!



Benchmark

Aneurysm geometry provided by R. Cebal, GMU, USA

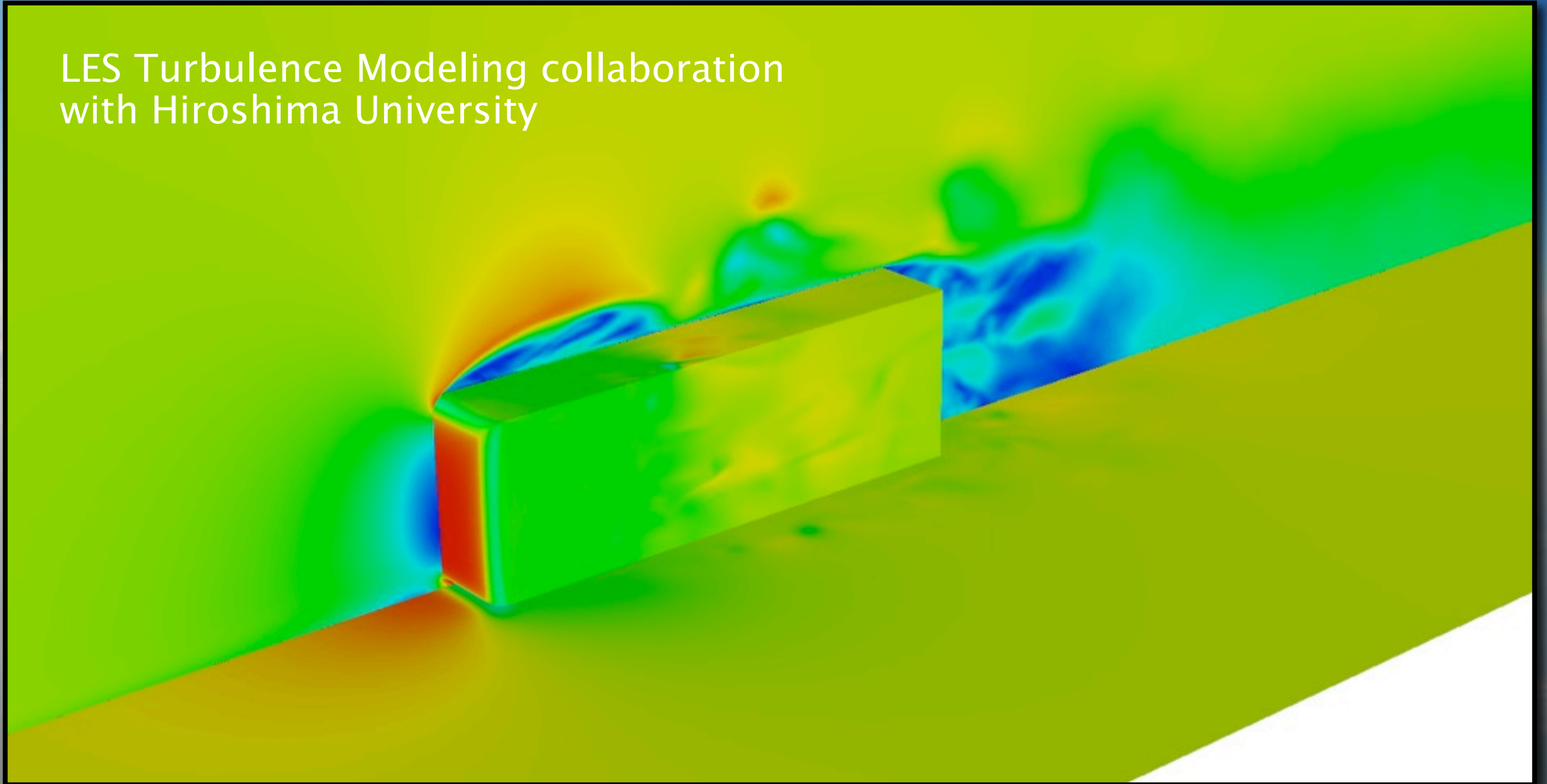
Uniform refinement up to 1.6 Billion tetrahedra

Incompressible flow

Implicit formulation

Algebraic Fractional Step: BCGStab + Deflated CG

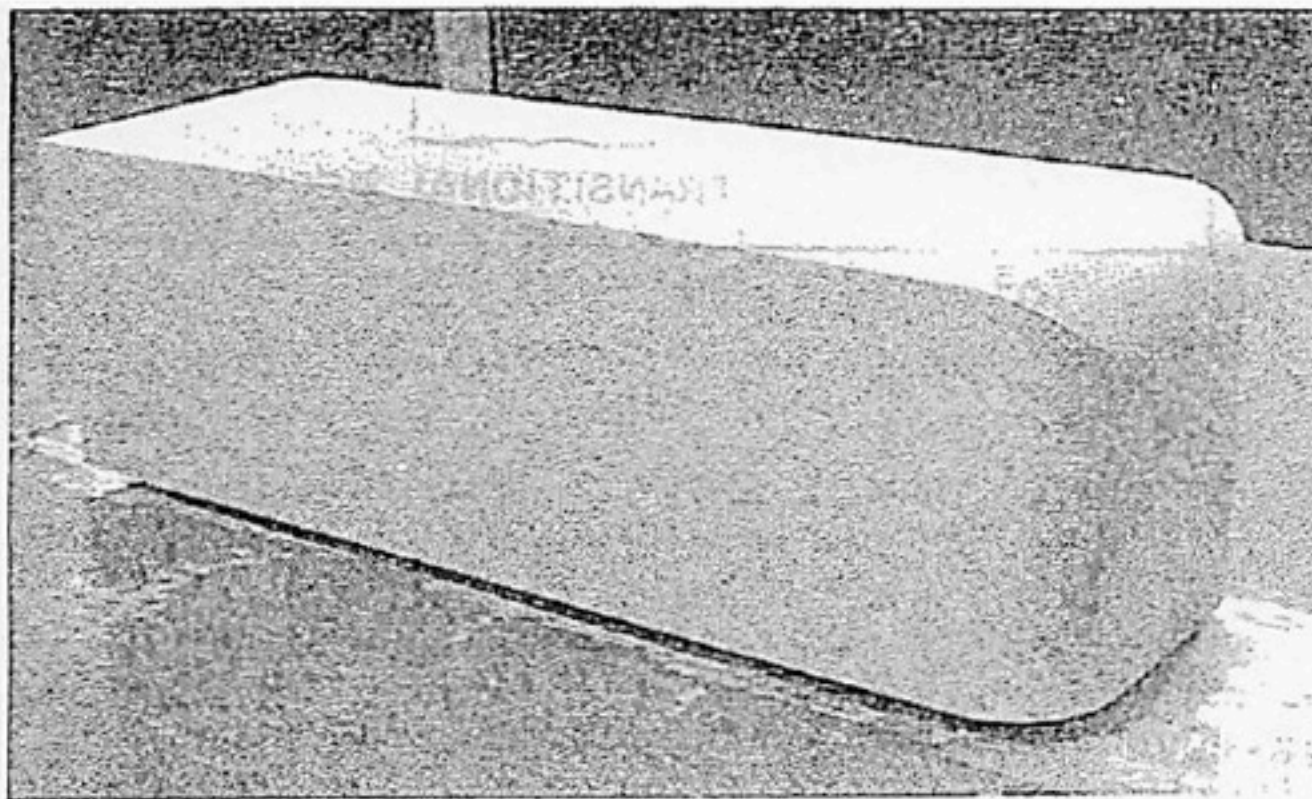
LES Turbulence Modeling collaboration  
with Hiroshima University





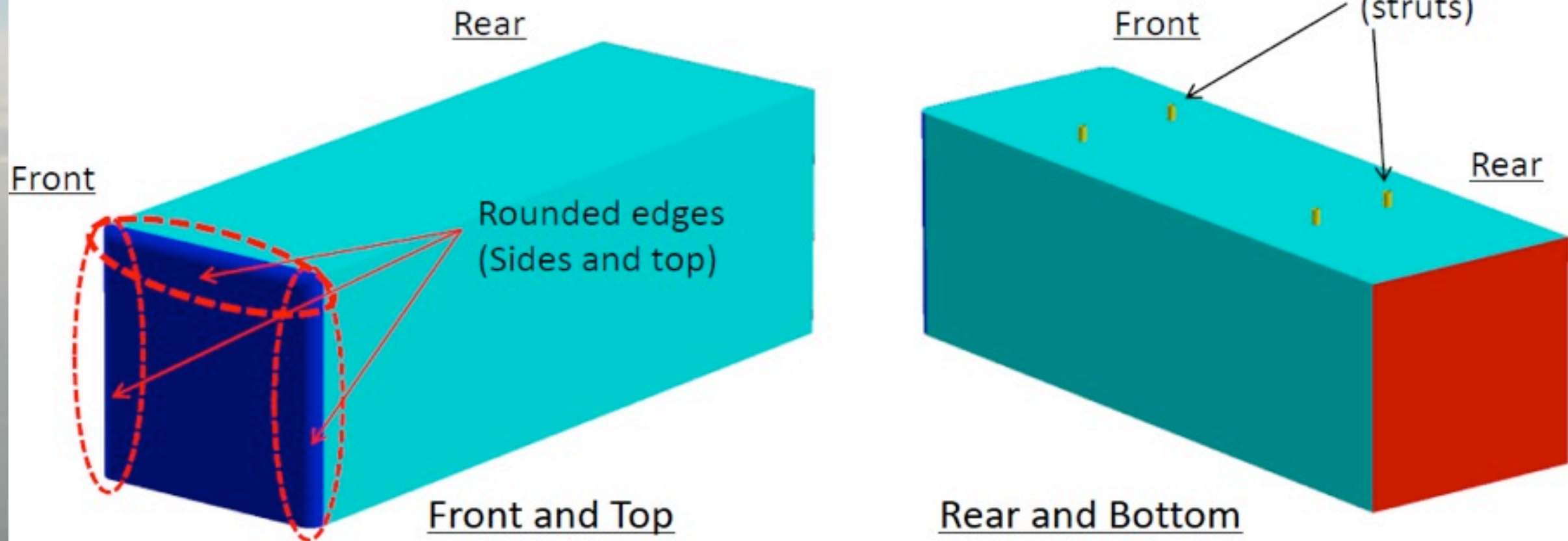
## Reference paper

- Cooper (1985)
  - “The Effect of Front-Edge Rounding and Rear-Edge Shaping on the Aerodynamic Drag of Bluff Vehicles in Ground Proximity”, SAE paper no. 850288, 1985.



## Objective model 1/2

- Box with front-edge rounding
  - Schematic views



- Size: Length  $L=1143\text{mm}$ , Width  $w=381\text{mm}$ , Height  $h=381\text{mm}$
- Curvature radius of the rounded front edge:  $\eta$

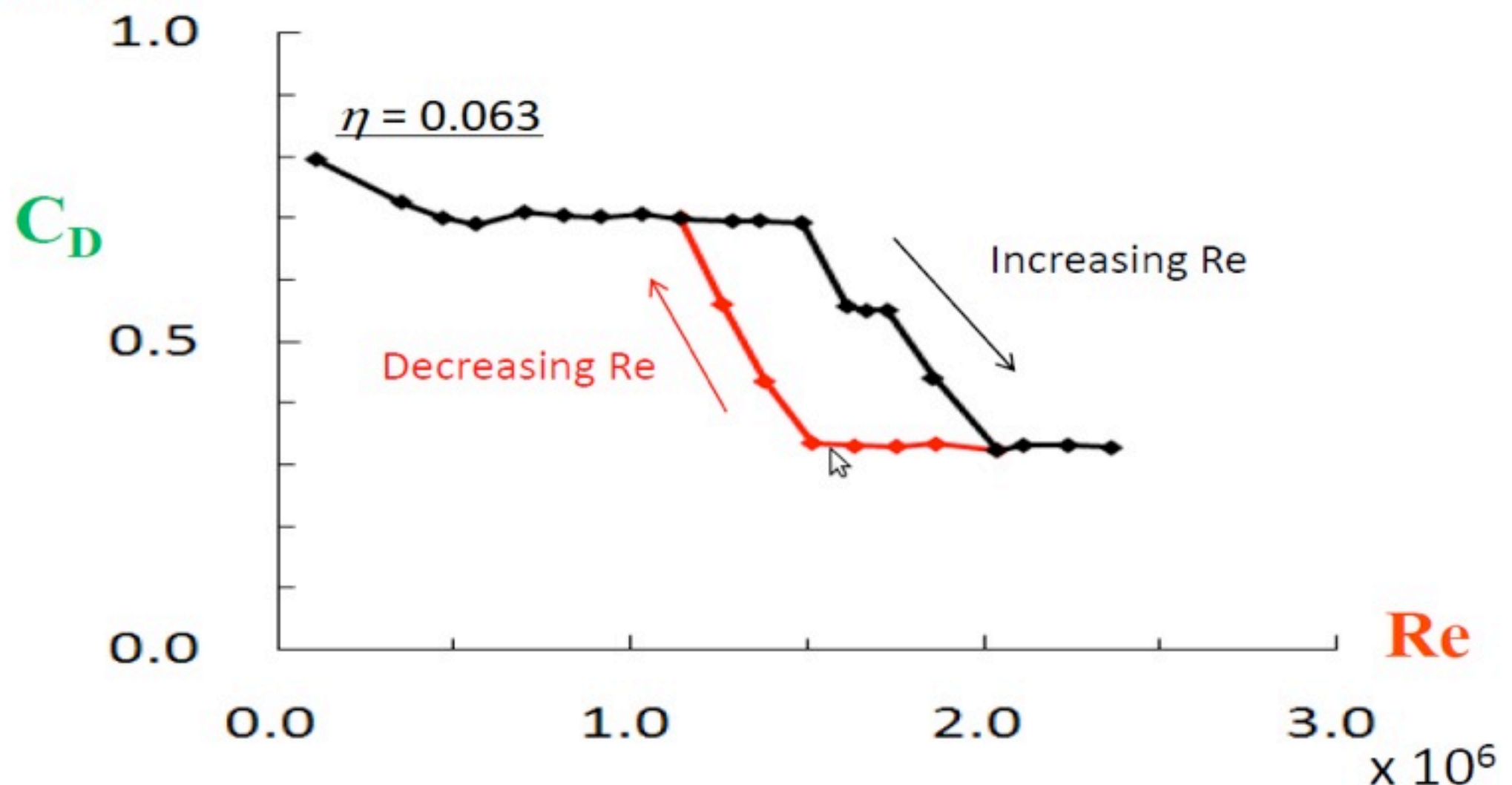


Interesting phenomenon

# Hysteresis of aerodynamics

- Cooper's result

– Digitized



## Surface flow in Experiment (Cooper, 1985)

THREE different flow structures around the body

$Re_A = 1.29 \times 10^6$   
( $U = 50 \text{ m/s}$ )  
 $C_D = 0.695$   
Flow Separation  
Top: Yes, Side: Yes

$Re_A = 1.67 \times 10^6$   
( $U = 60 \text{ m/s}$ )  
 $C_D = 0.550$   
Flow Separation  
Top: Yes, Side: No

$Re_A = 2.04 \times 10^6$   
( $U = 80 \text{ m/s}$ )  
 $C_D = 0.324$   
Flow Separation  
Top: No, Side: No

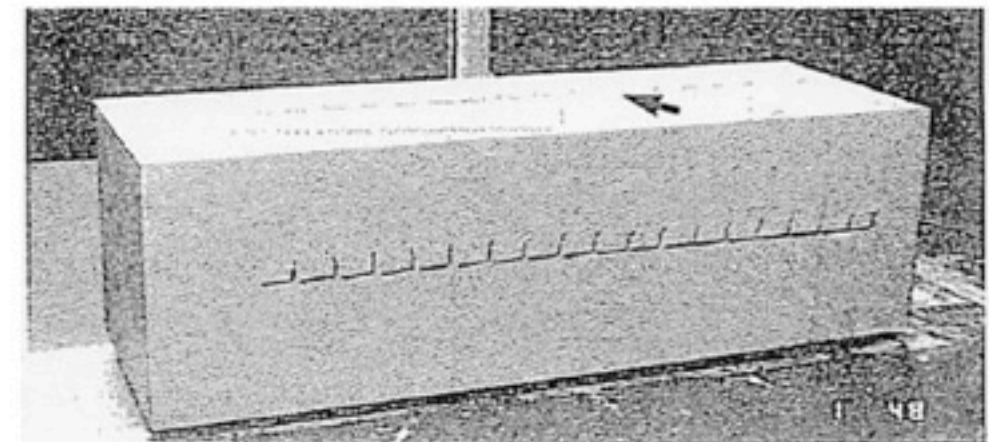
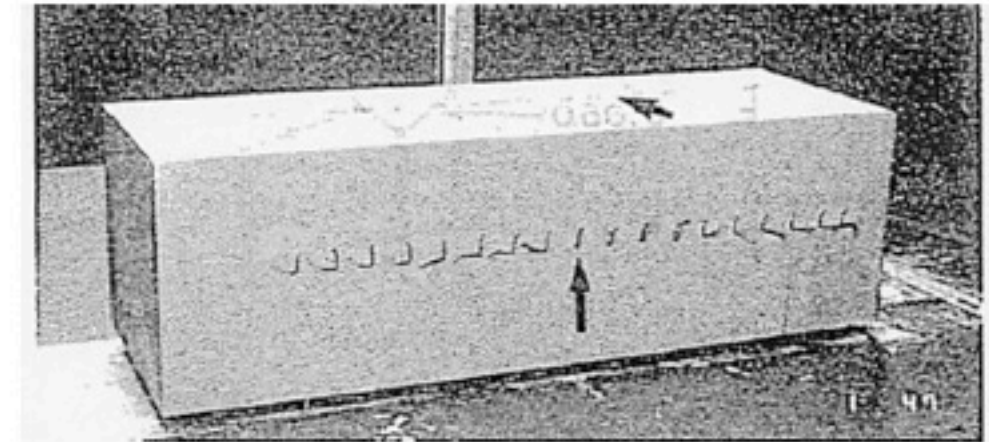
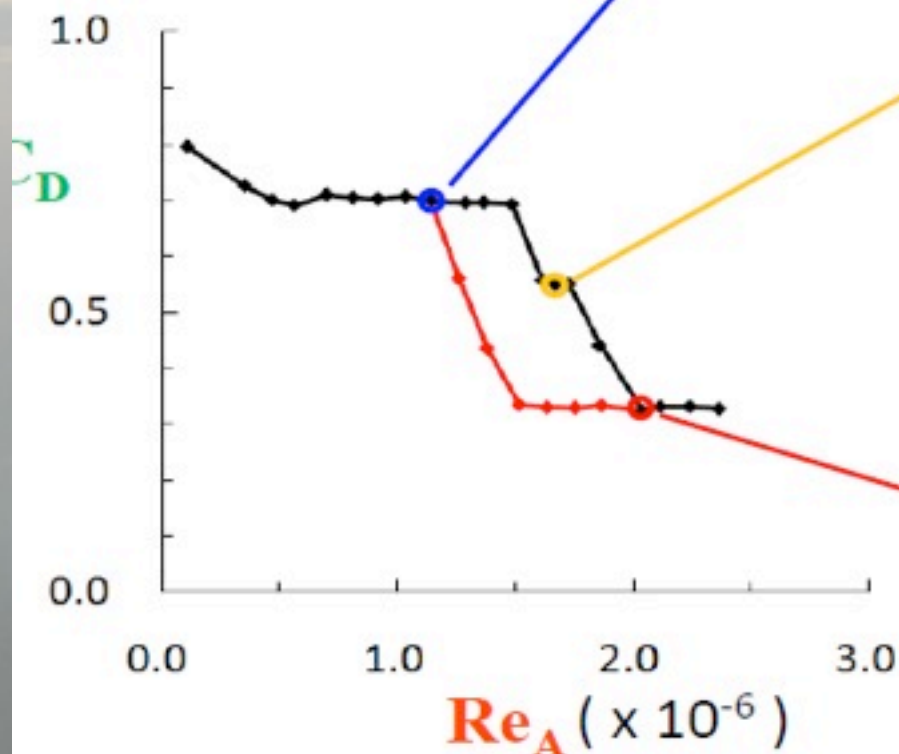


FIG. 16: SURFACE FLOW AT  $\psi = 0^\circ$  WITH  $\eta = 0.063$   
(REATTACHMENT POINTS INDICATED BY ARROWS)



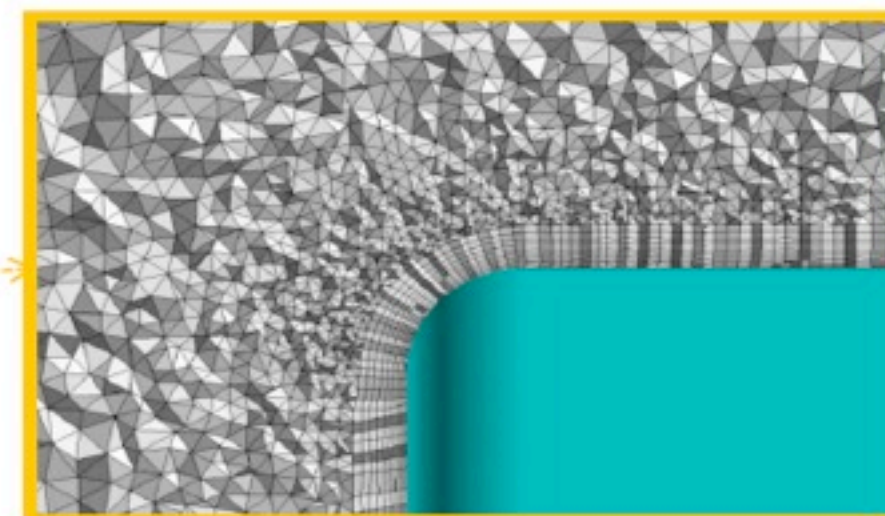
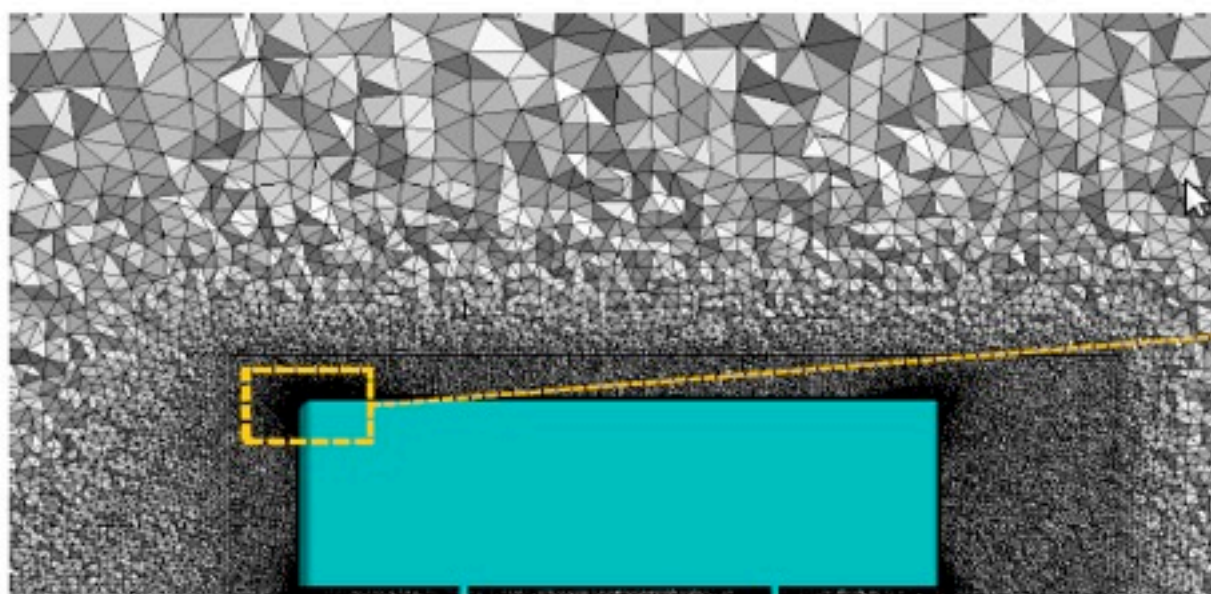
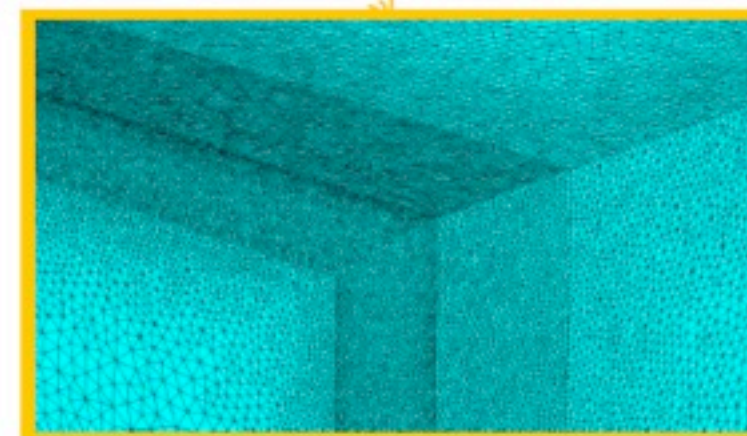
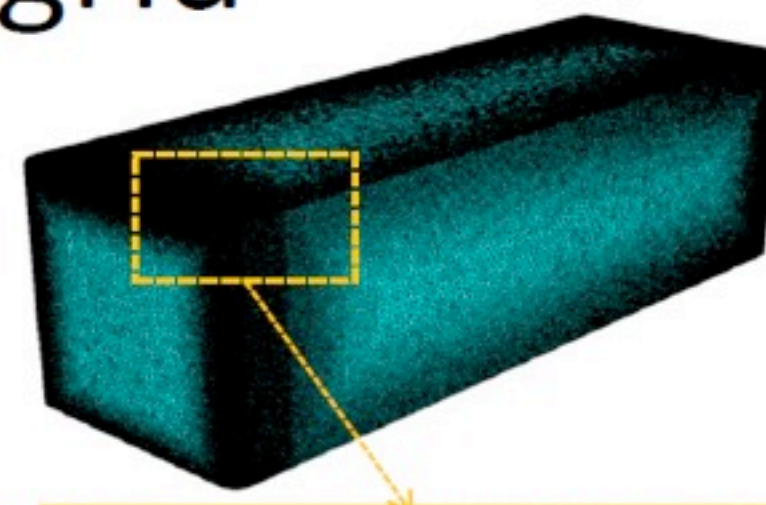
## NUMERICAL AND PHYSICAL MODEL

- Wall-adapting local eddy-viscosity (WALE) model
- Stabilized Finite element method with tracking of sub-scales (evolution of SUPG and GLS techniques).
- Collaboration with professor Takuji Nakashima from Hiroshima University (Japan).

# Computational grid

- Unstructured grid

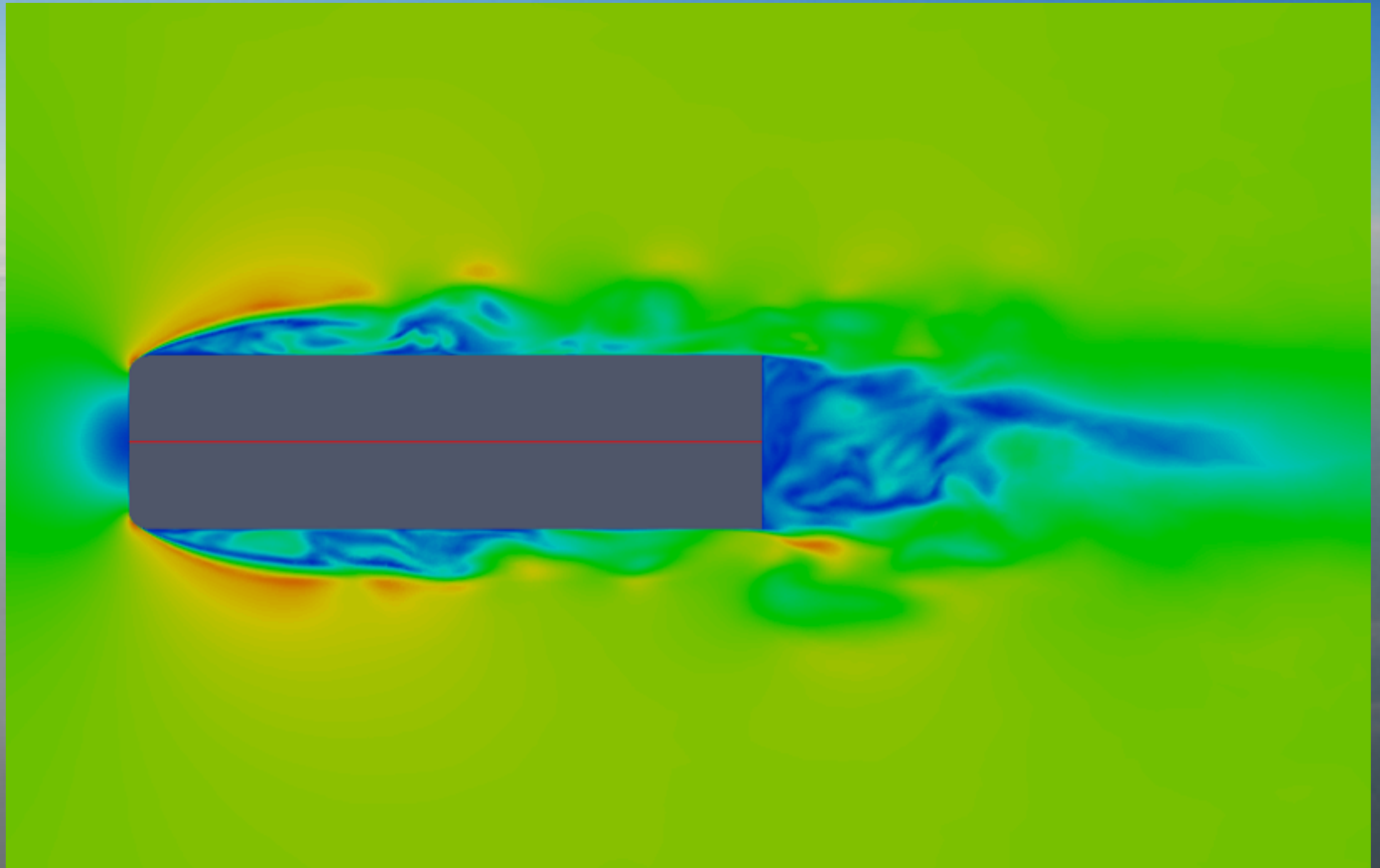
NODAL\_POINTS= 6409847  
ELEMENTS= 20387394





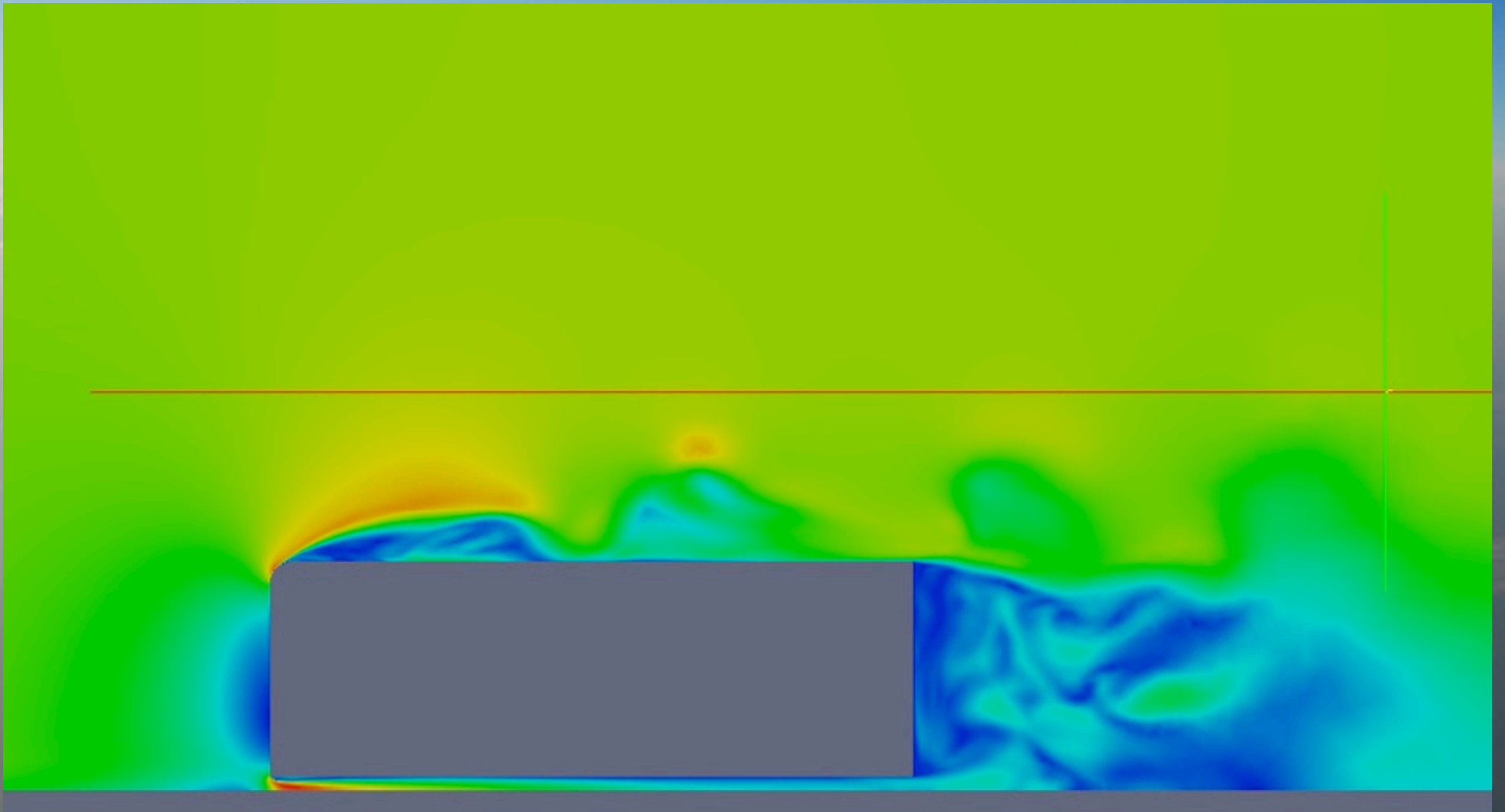
## Numerical results

Velocity Top view



## Numerical results

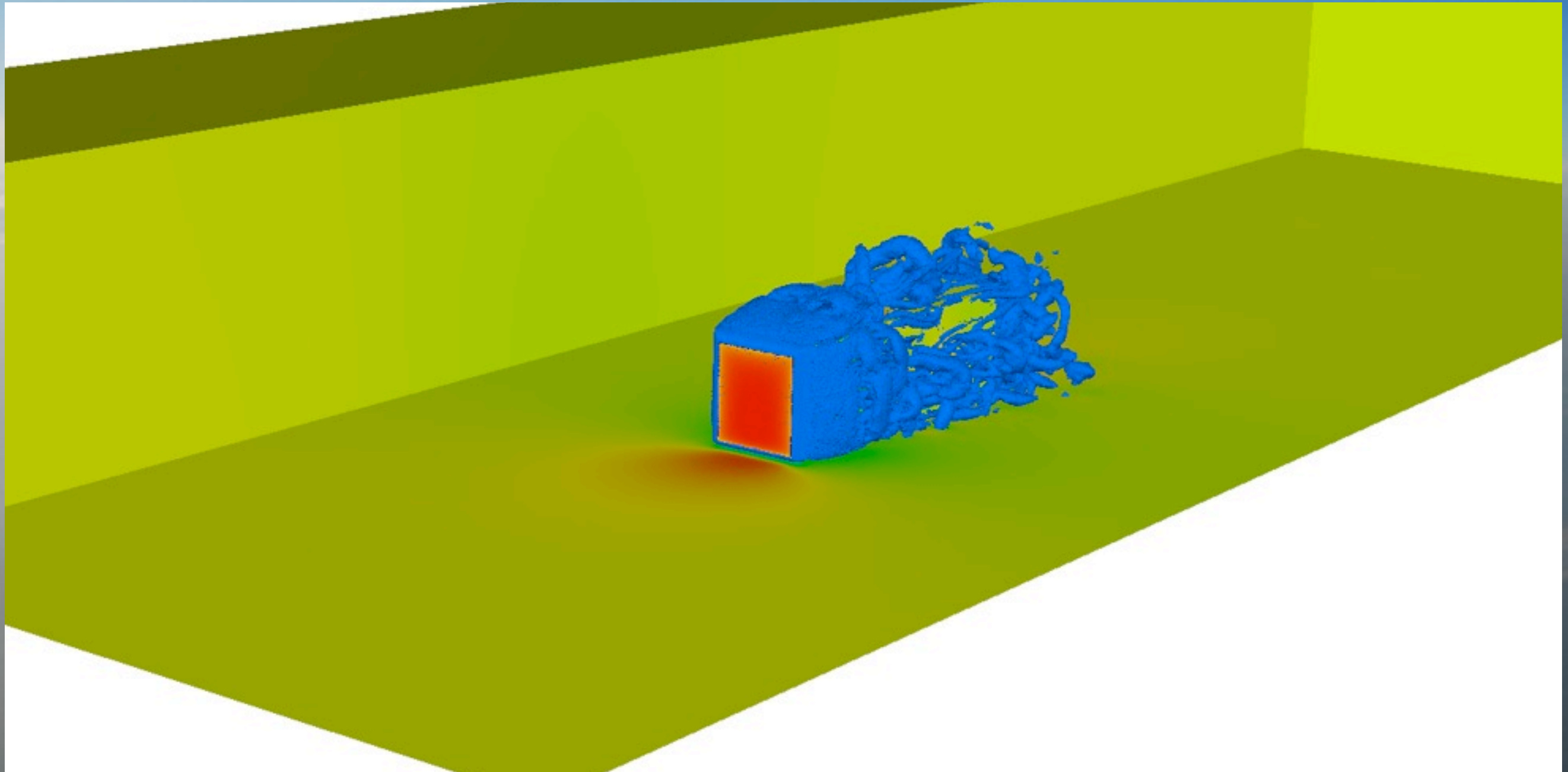
### Velocity Lateral view



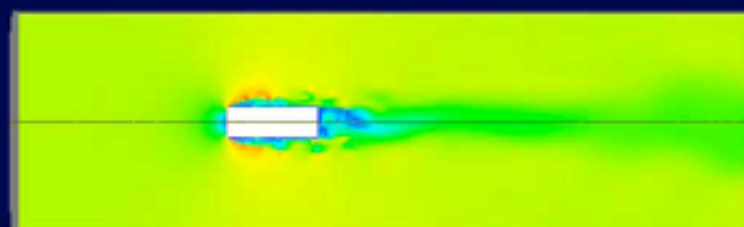


## Numerical results

Qvorticity



## Numerical results - Velocity



norm(\$4,\$5,\$6)

6

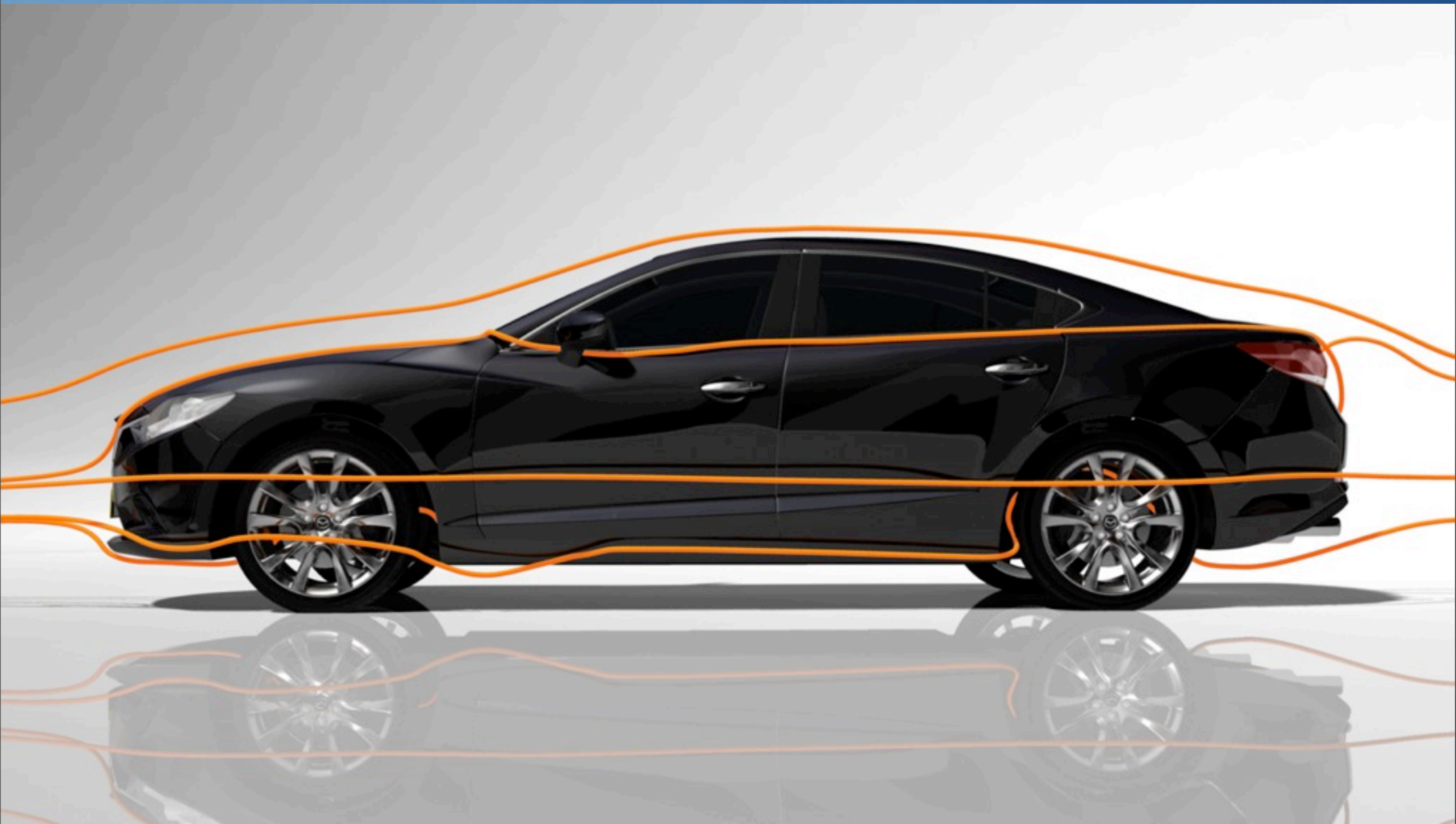
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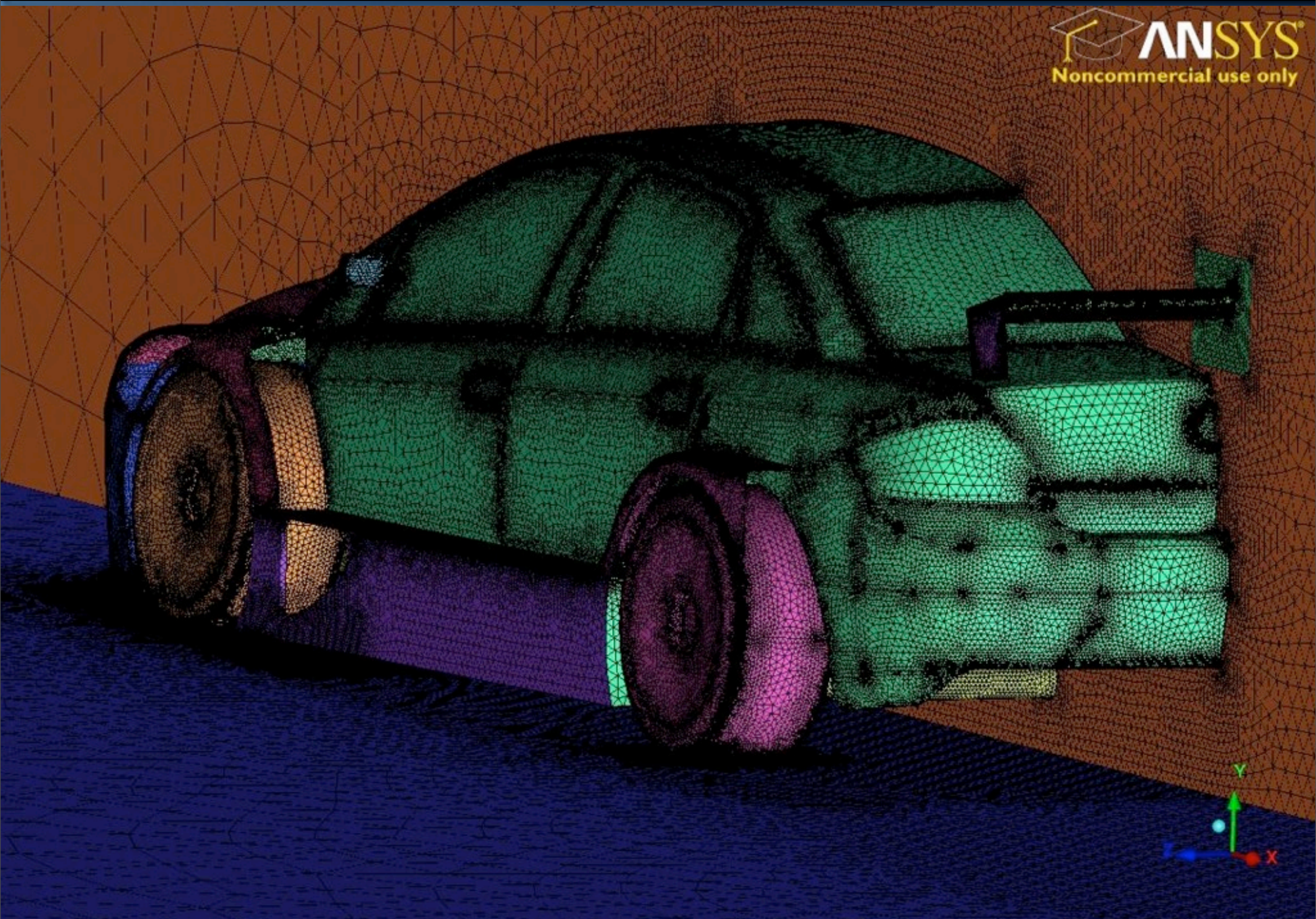
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30

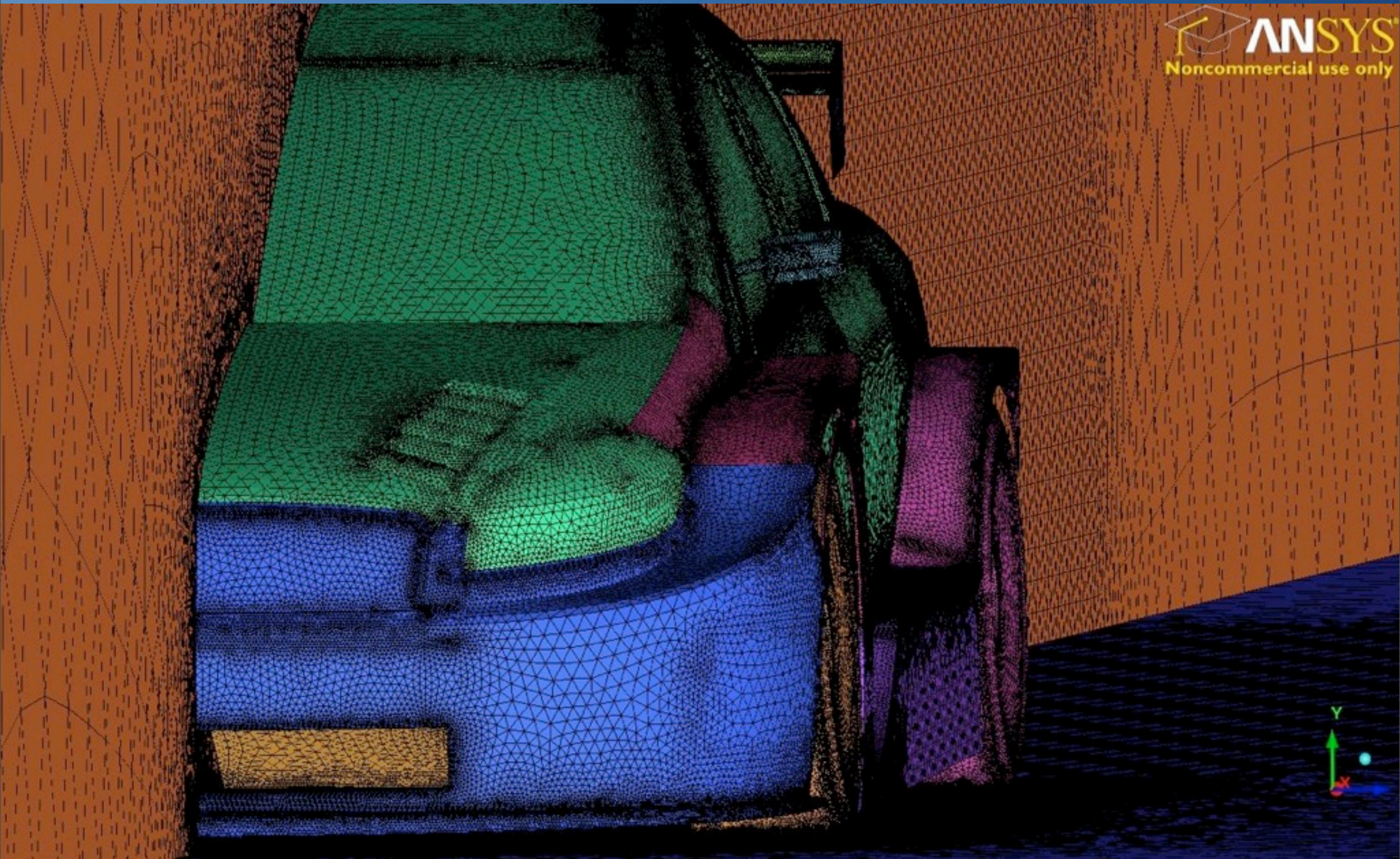




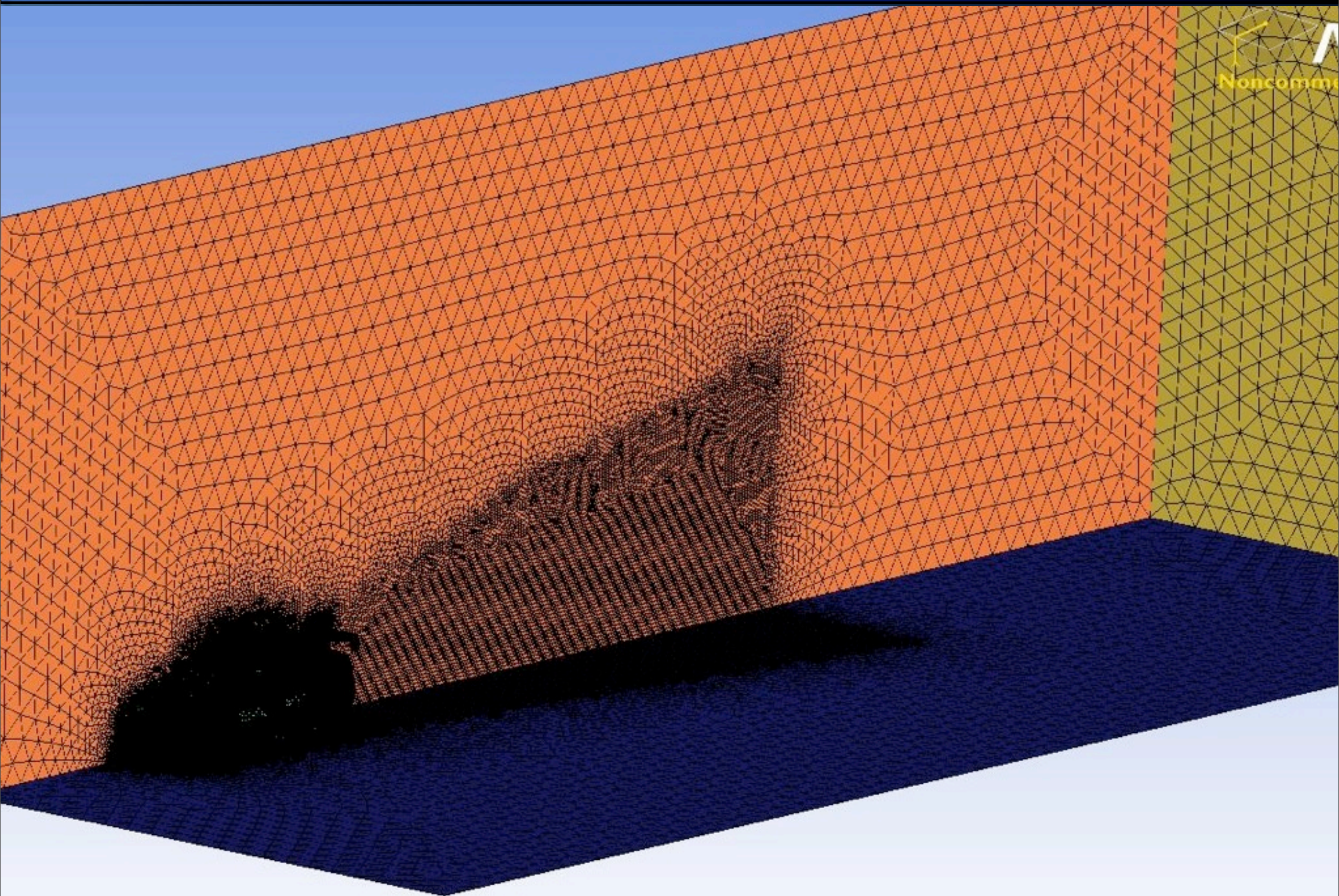








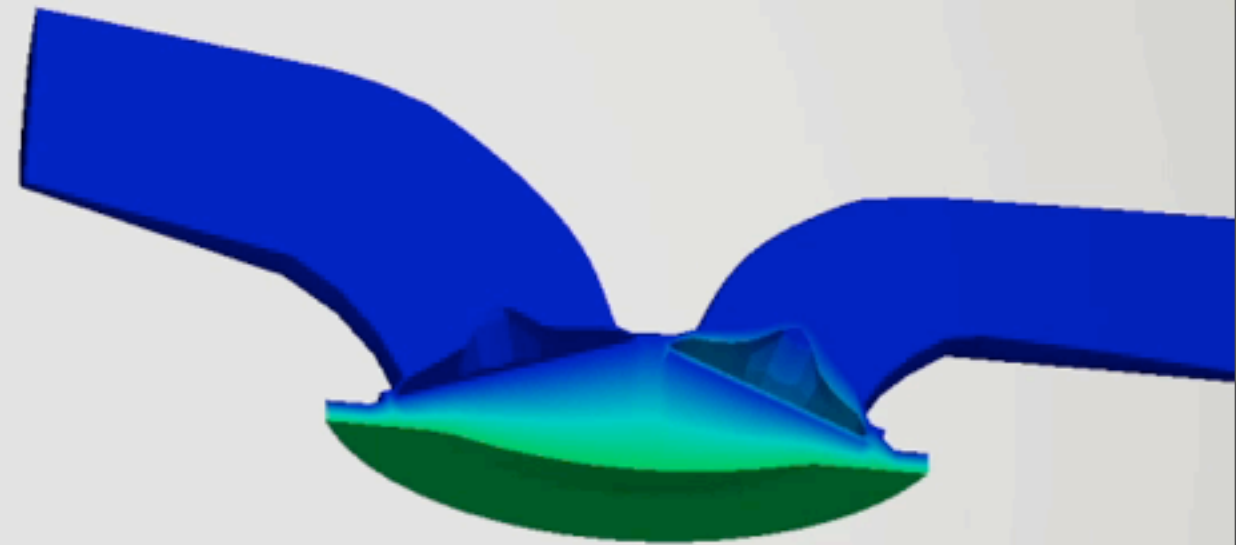
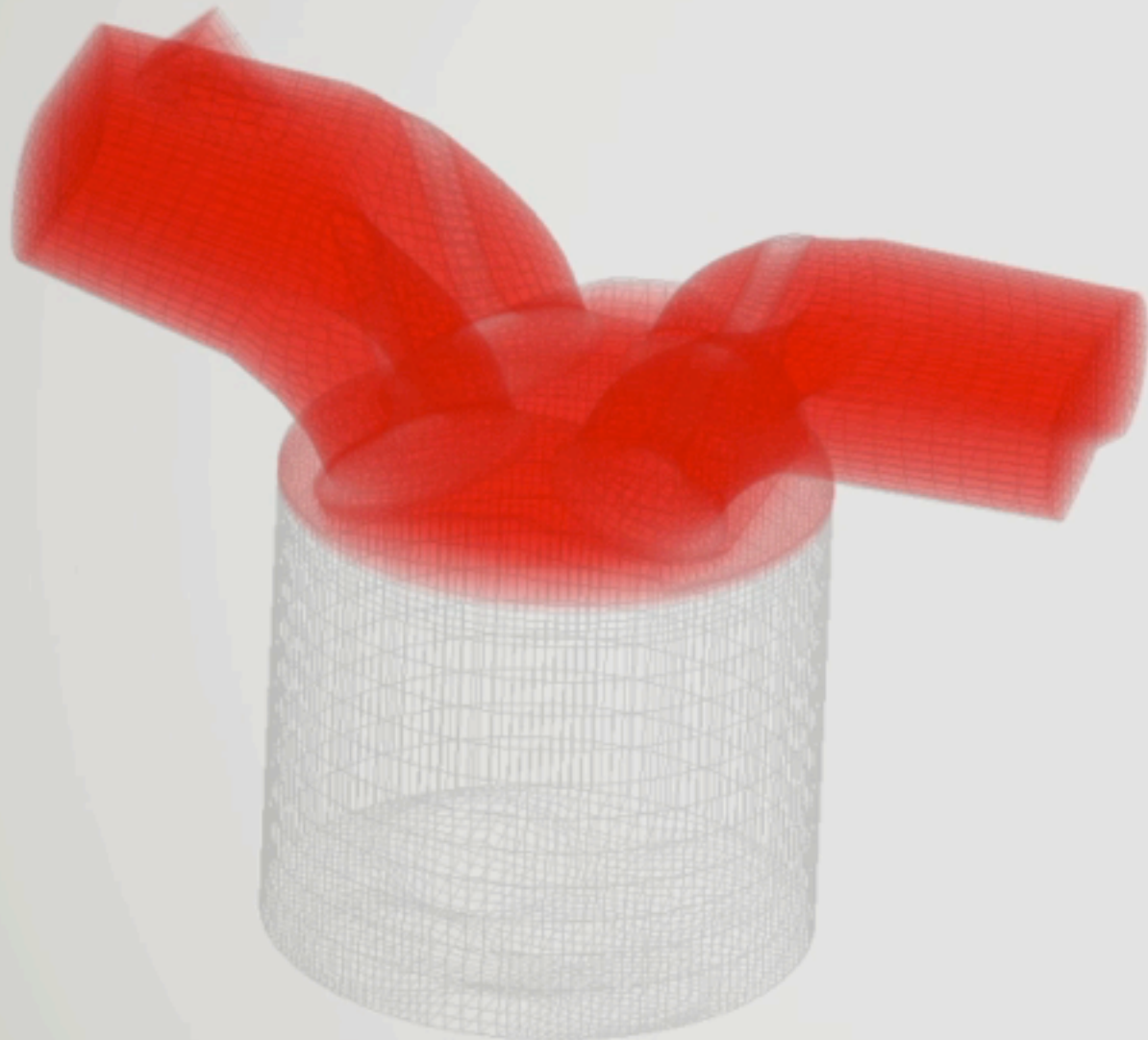








## 4 valves engine - RNG k-epsilon



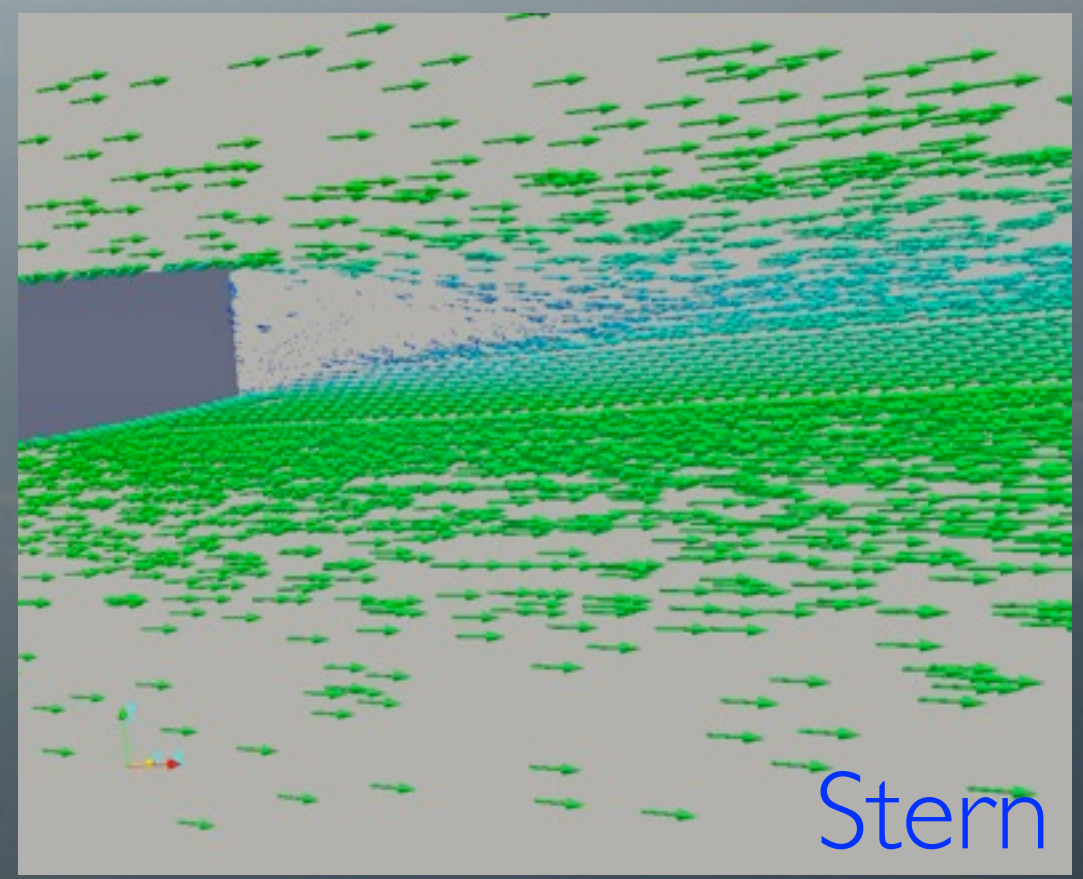
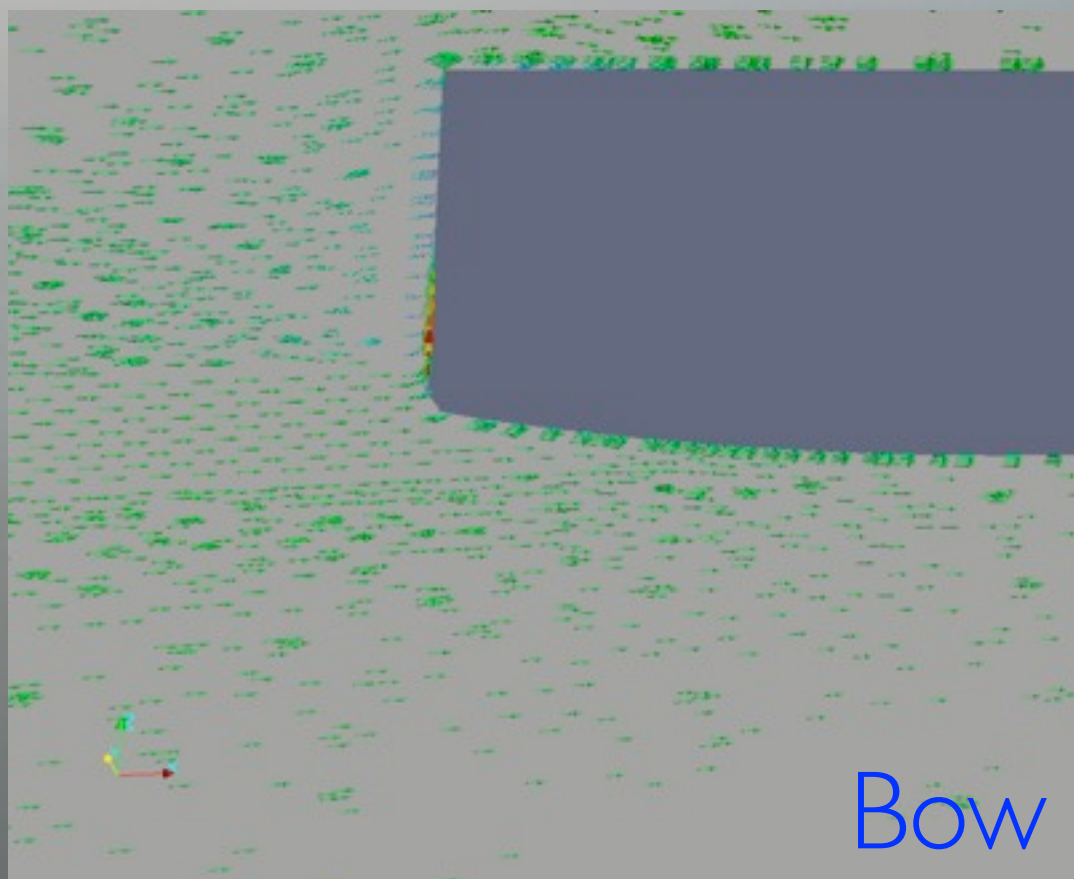
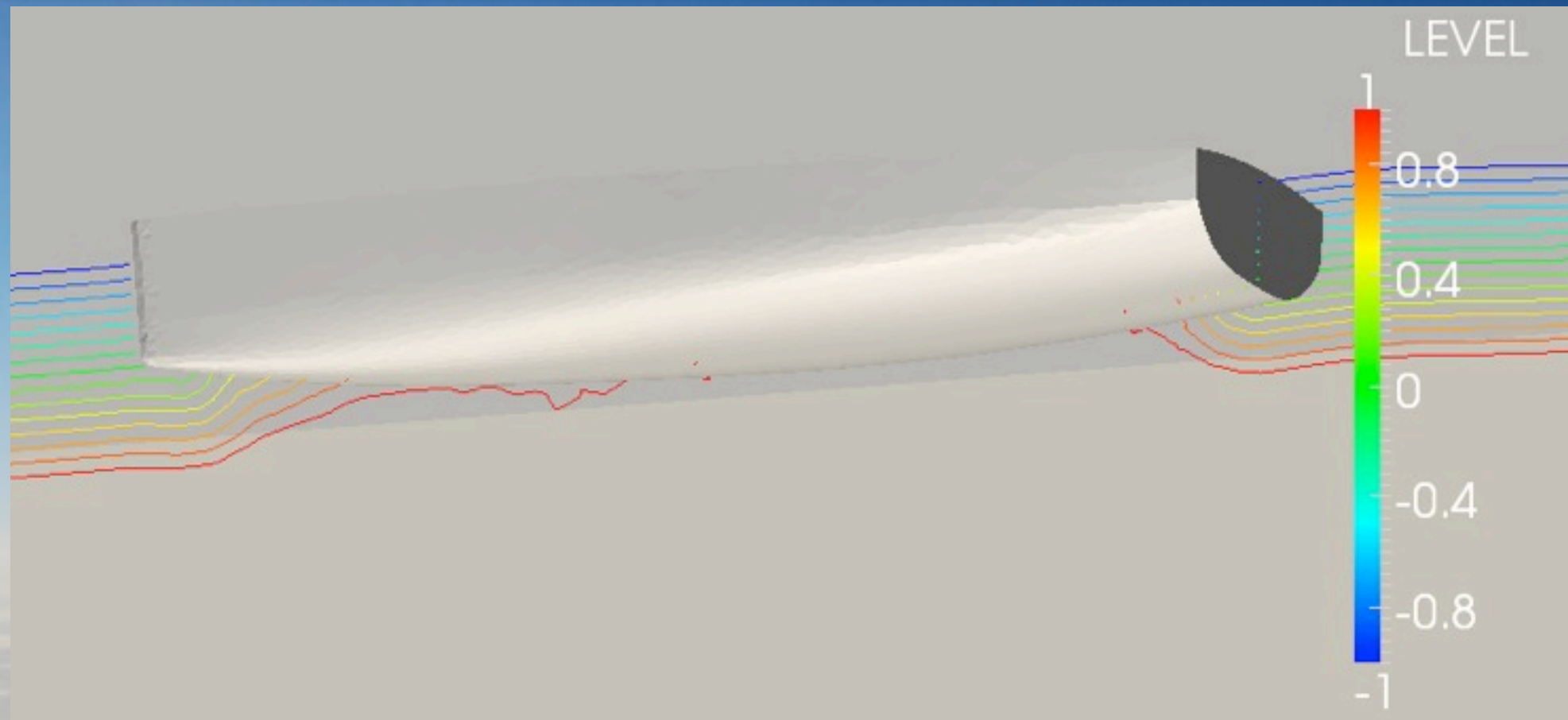
## Physical and Numerical Model

- K-omega SST Turbulence model
- Level set method to capture the position of the interface
- Stabilized Finite element method to solve Navier Stokes, Level Set and turbulence equations.

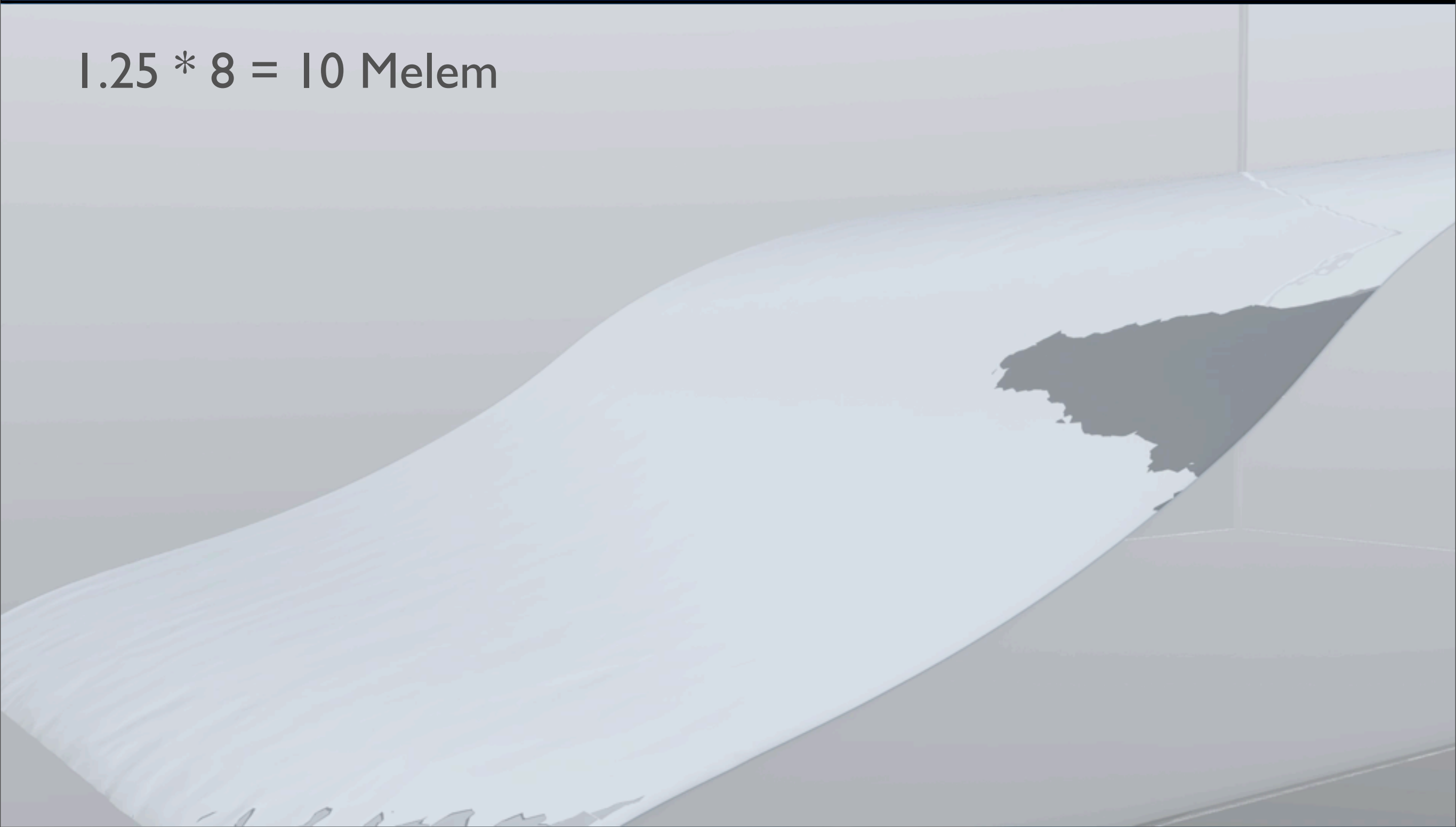
## 2 Fluid Flow

- Level Set
- Smoothed properties.
- Geometrical redistancing using kdtree.





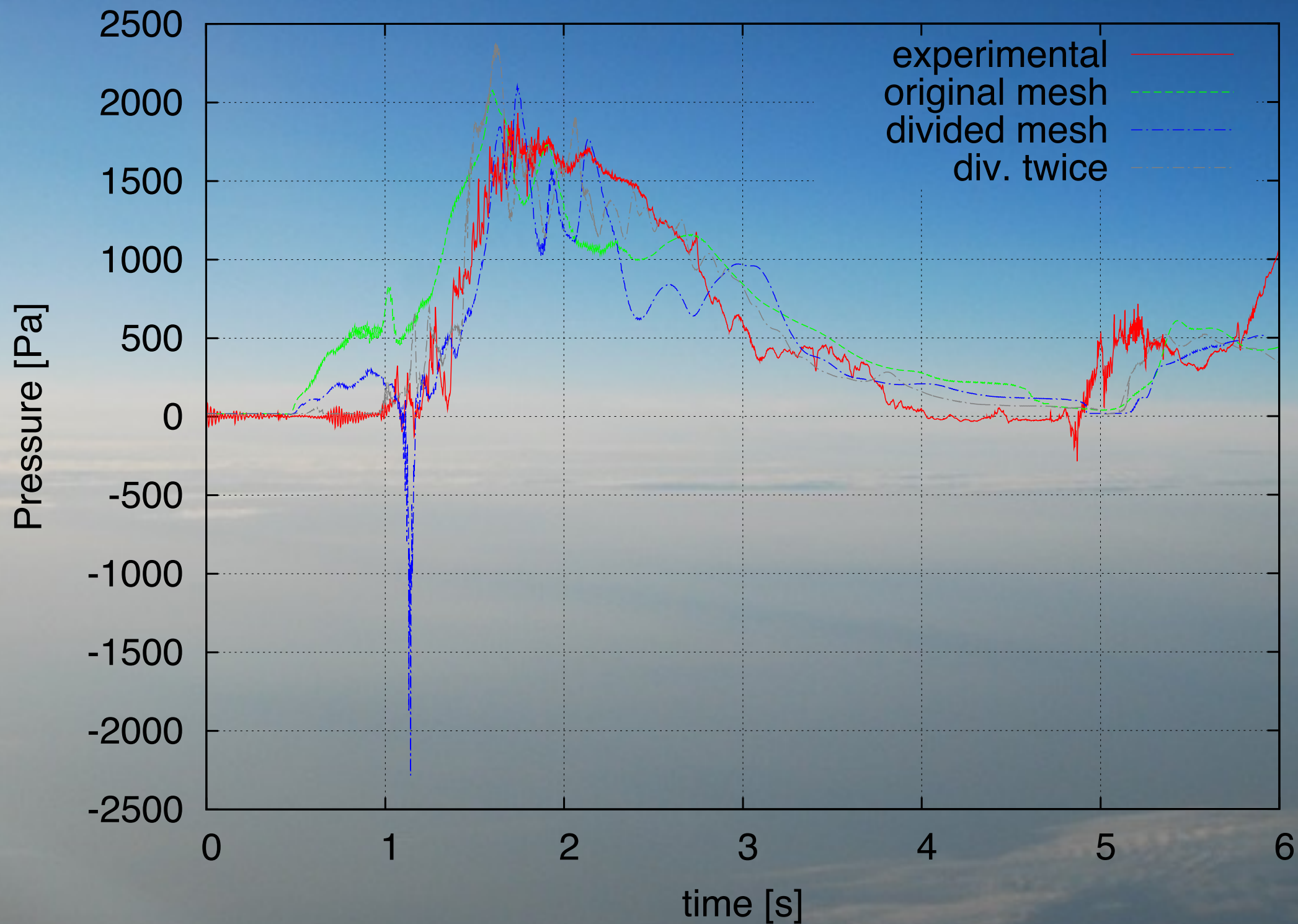
$$1.25 * 8 = 10 \text{ Melem}$$



Green Water Dam Break (1 div)

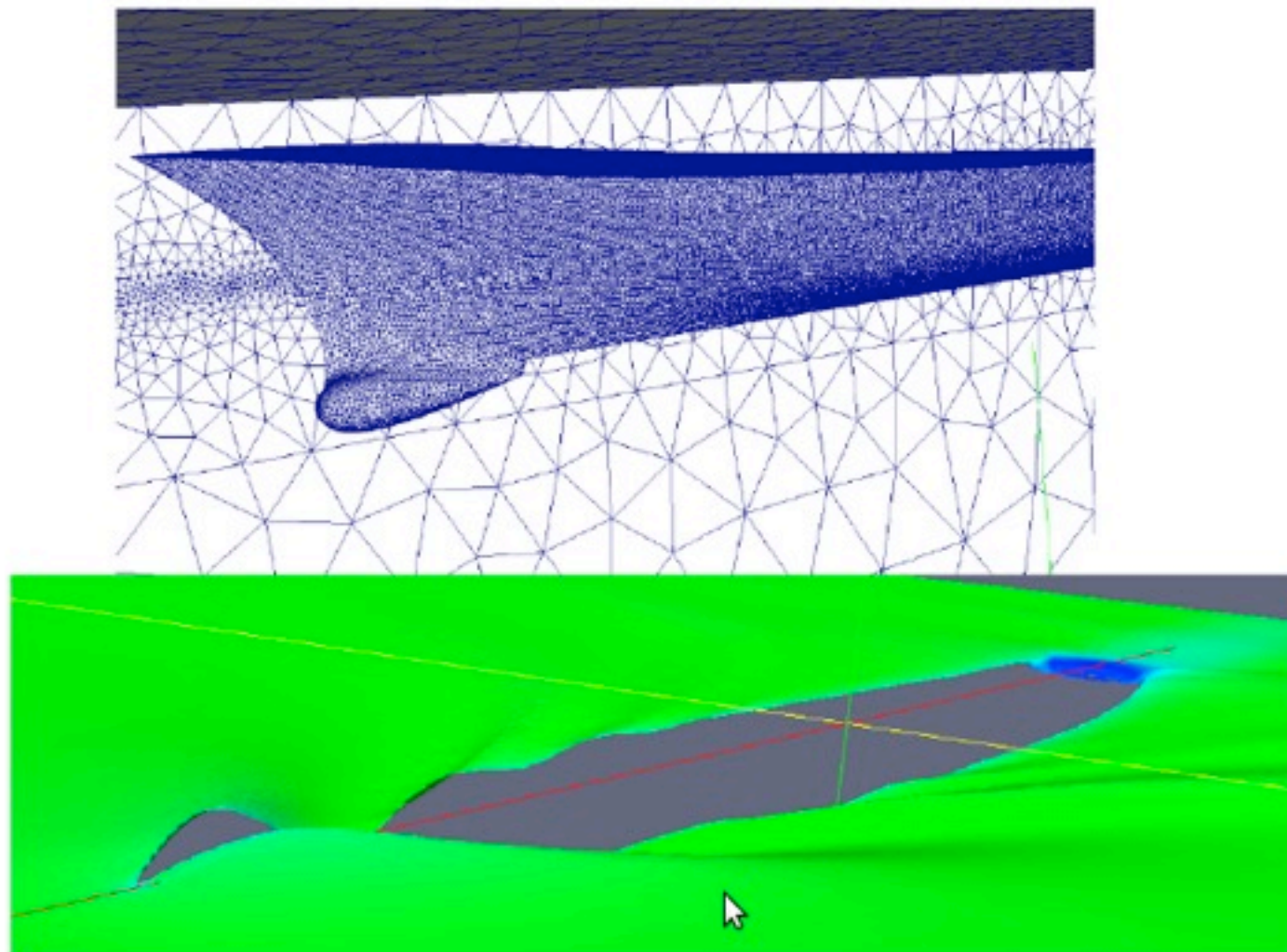
<http://wiki.manchester.ac.uk/spheric/index.php>





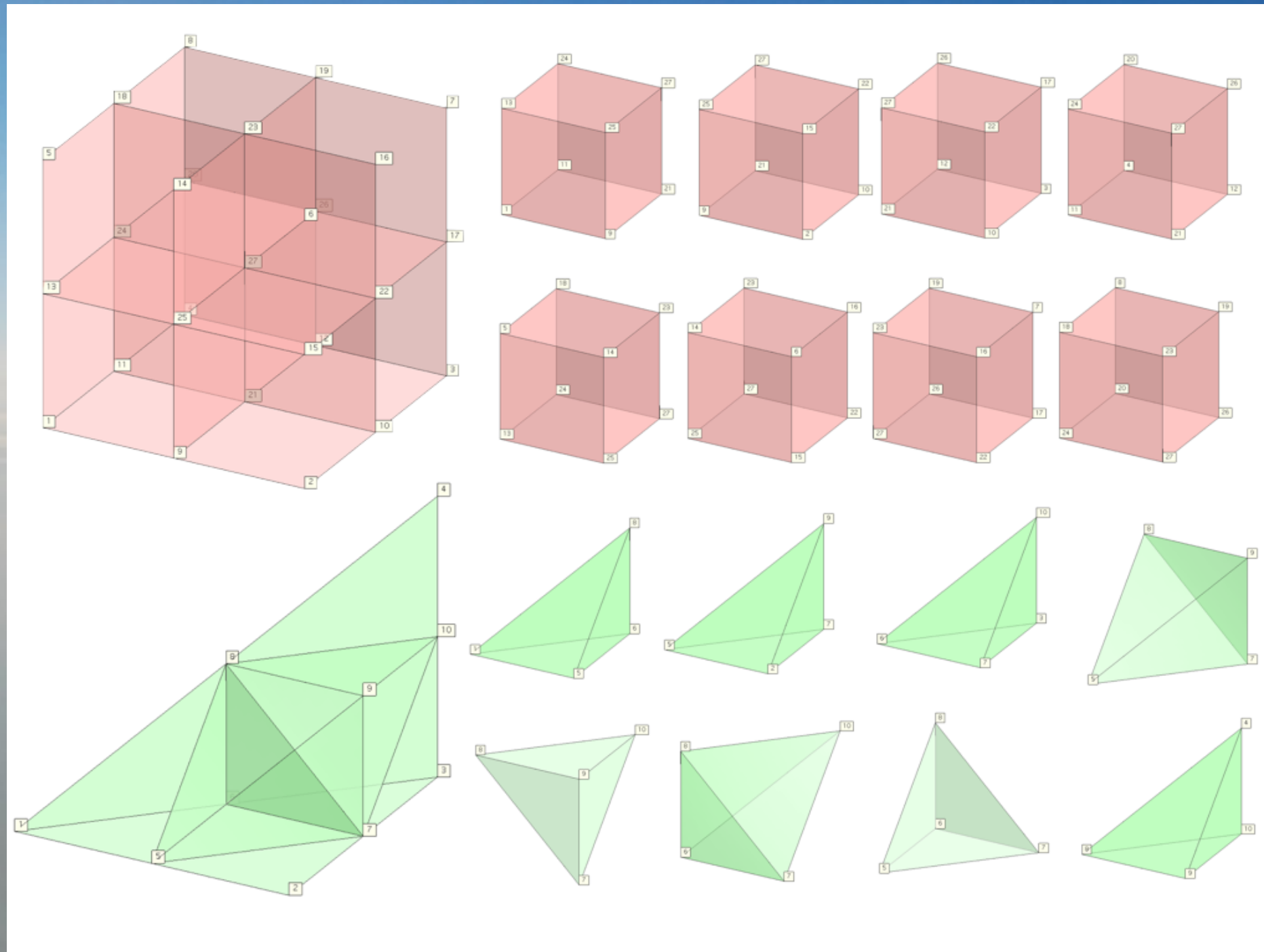
Green Water Dam Break vs. experimental (P6)

## Free surface shape and meshes used



- Mesh A: 8Melem (7.5 tetrahedras + 0.5 prisms).
- Mesh B: 5Melem (4.6 tetrahedras + 0.4 prisms). Half domain - Symmetry.
- Mesh C: 40Melem. Previous mesh divided into elements with half the size.





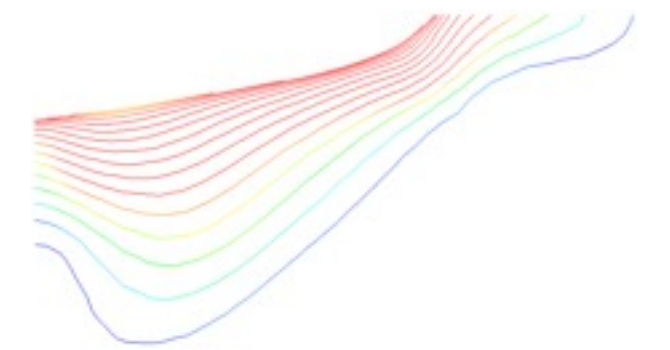
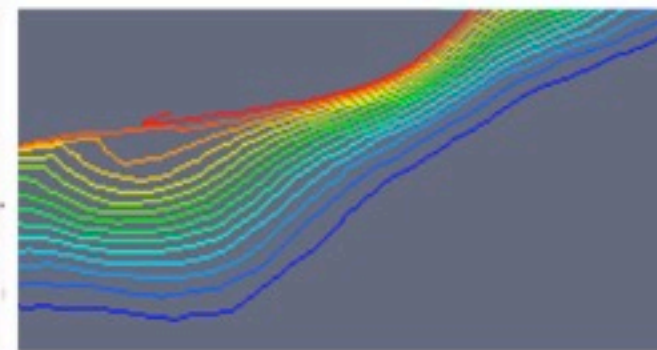
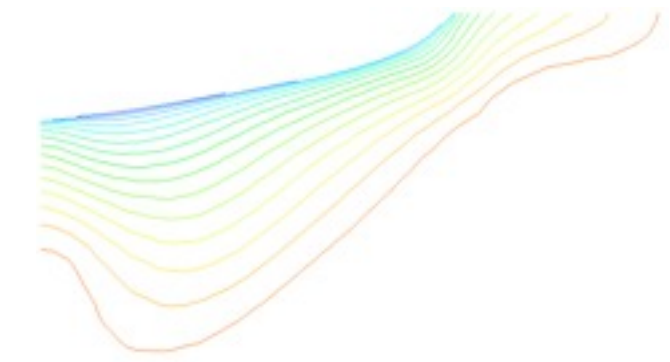
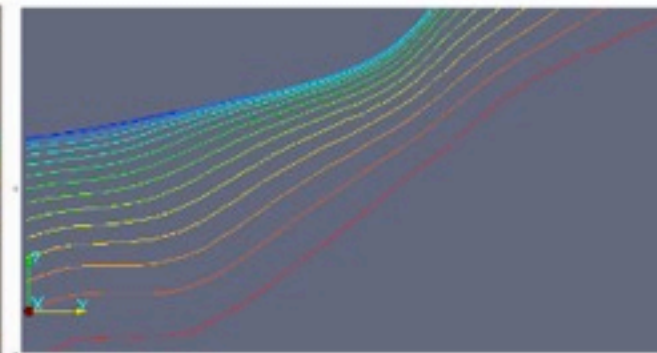
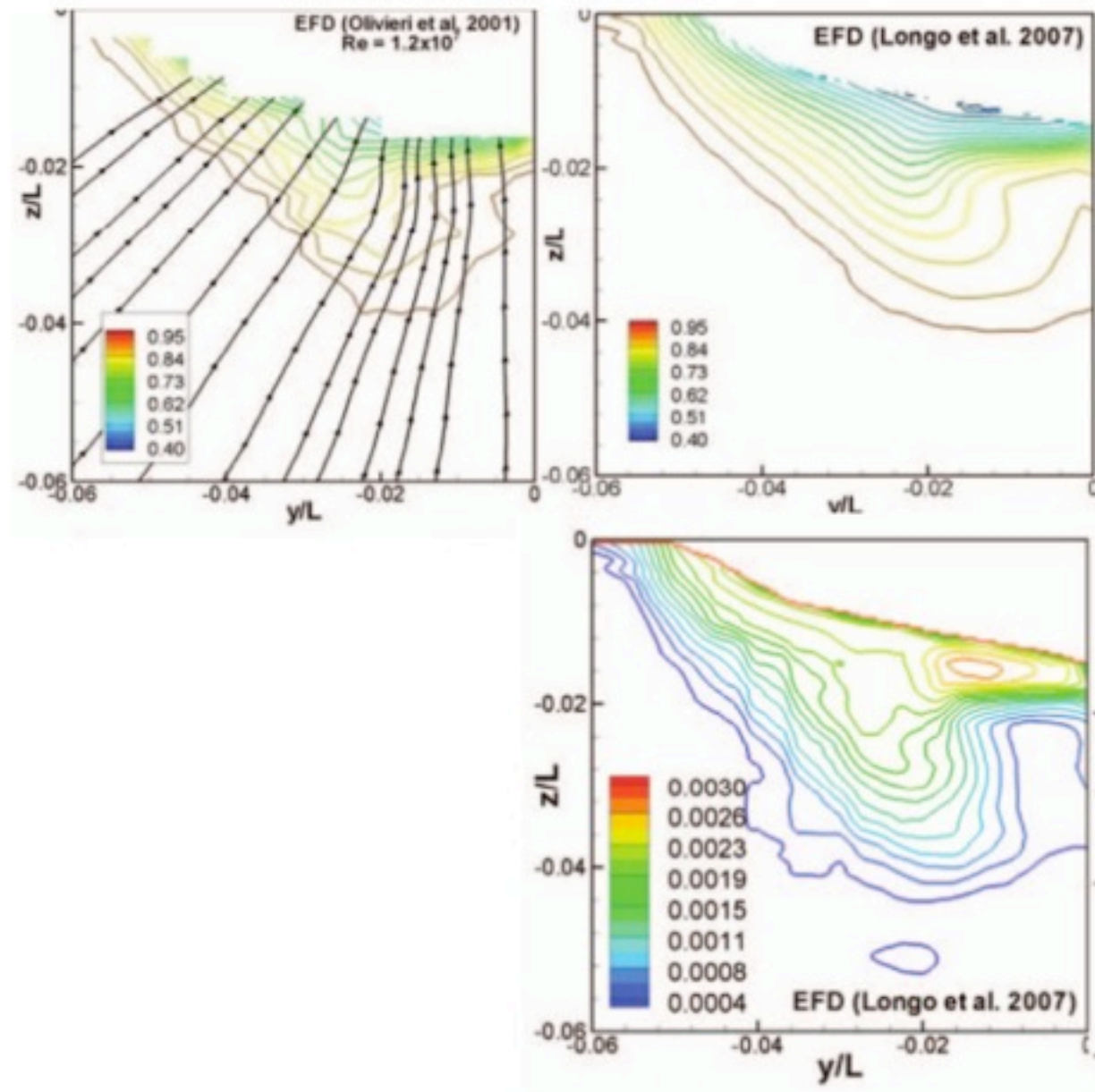
Guillaume Houzeaux, Raúl de la Cruz, Herbert Owen, and Mariano Vázquez. Parallel uniform mesh multiplication applied to a Navier- Stokes solver. *Computers and Fluids*

## Benchmarking

- David Taylor Model Basin model - DTMB 5512.
- 1:46.6 model scale of a modern surface combatant
- It has been tested in the towing tanks at DTMB, IIHR(Iowa) and INSEAN (Italy)



## Velocity and turbulent kinetic energy comparison



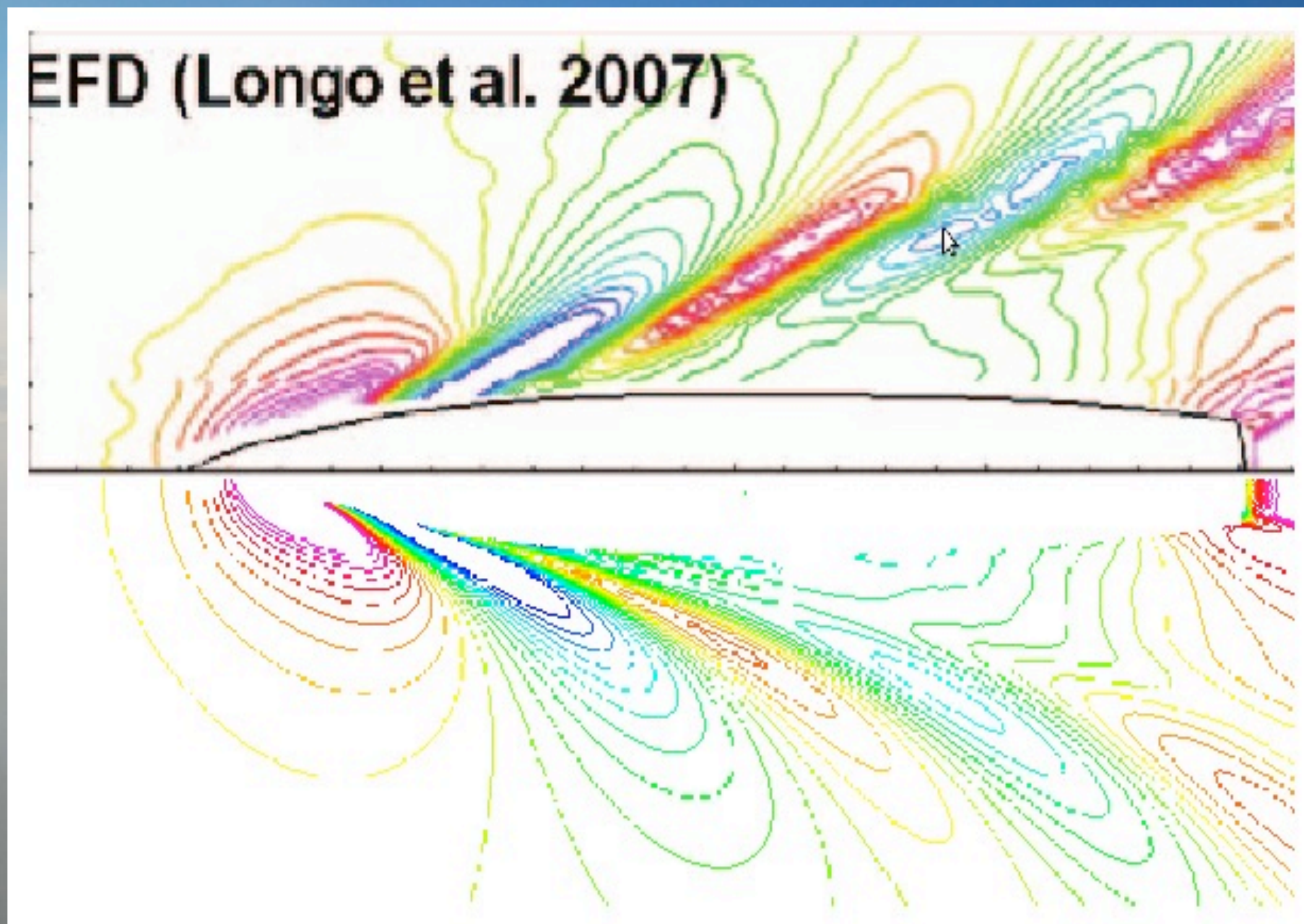
Original

Divided

Results at  $x/L=0.95$ . from: J. Longo, J. Shao, M. Irvine, and F. Stern. Phase-averaged piv for the nominal wake of a surface ship in regular head waves. Journal of Fluids Engineering, 129:524–540, 2007.

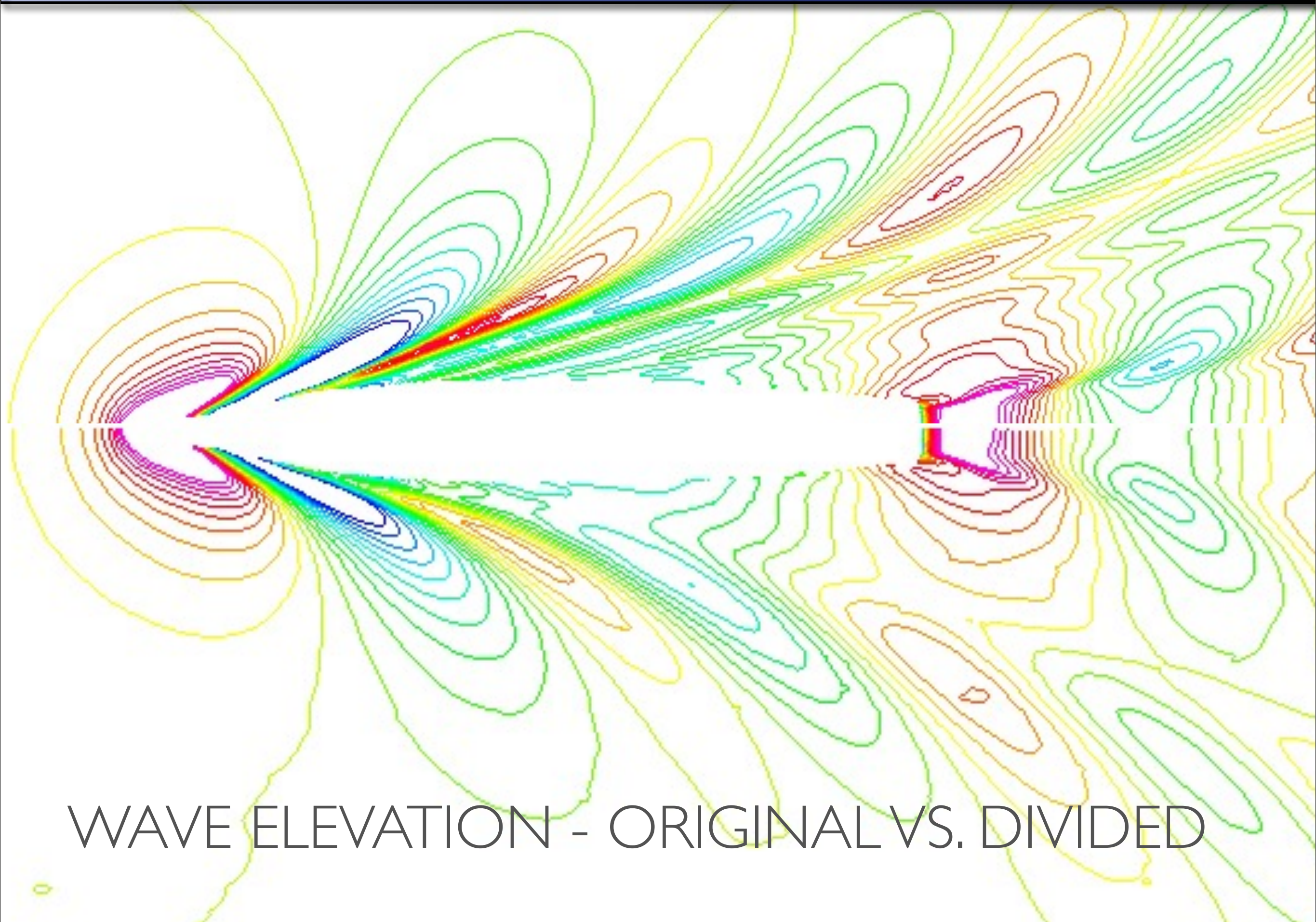


## Wave elevation - comparison with experimental results



Non dimensional wave elevation profiles for the 5512 hull. Top: experimental data, bottom: numerical result





WAVE ELEVATION - ORIGINAL VS. DIVIDED



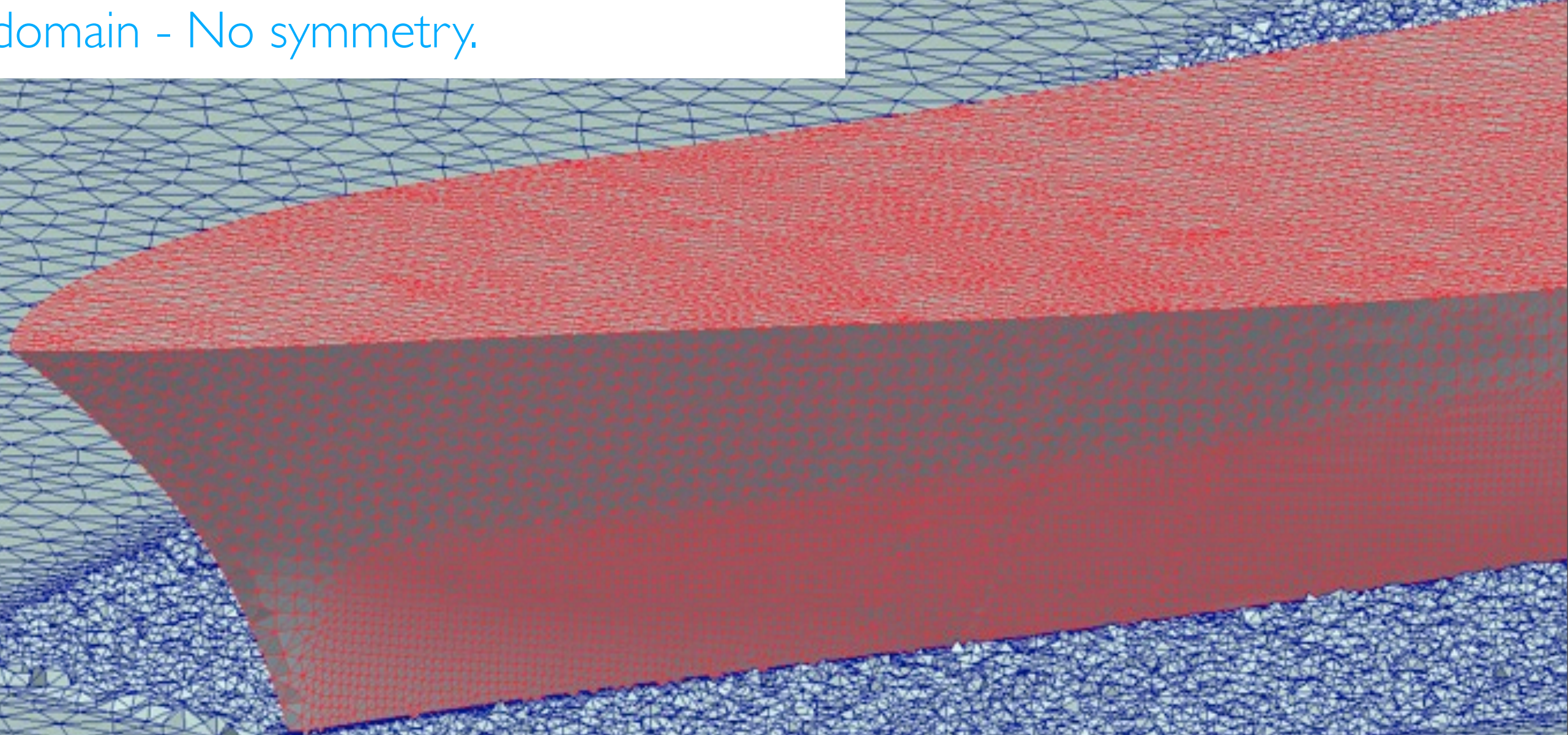
## Forces on the hull $Fr = 0.28$

|                        | Experimental            | Original Mesh | Divided Mesh |
|------------------------|-------------------------|---------------|--------------|
| $F_x[N]$ (total)       | 7.432                   | 7.41          | 7.30         |
| $F_{vx}[N]$ (viscous)  | 5.52 (from correlation) | 5.54          | 5.54         |
| $F_{px}[N]$ (pressure) |                         | 1.87          | 1.76         |



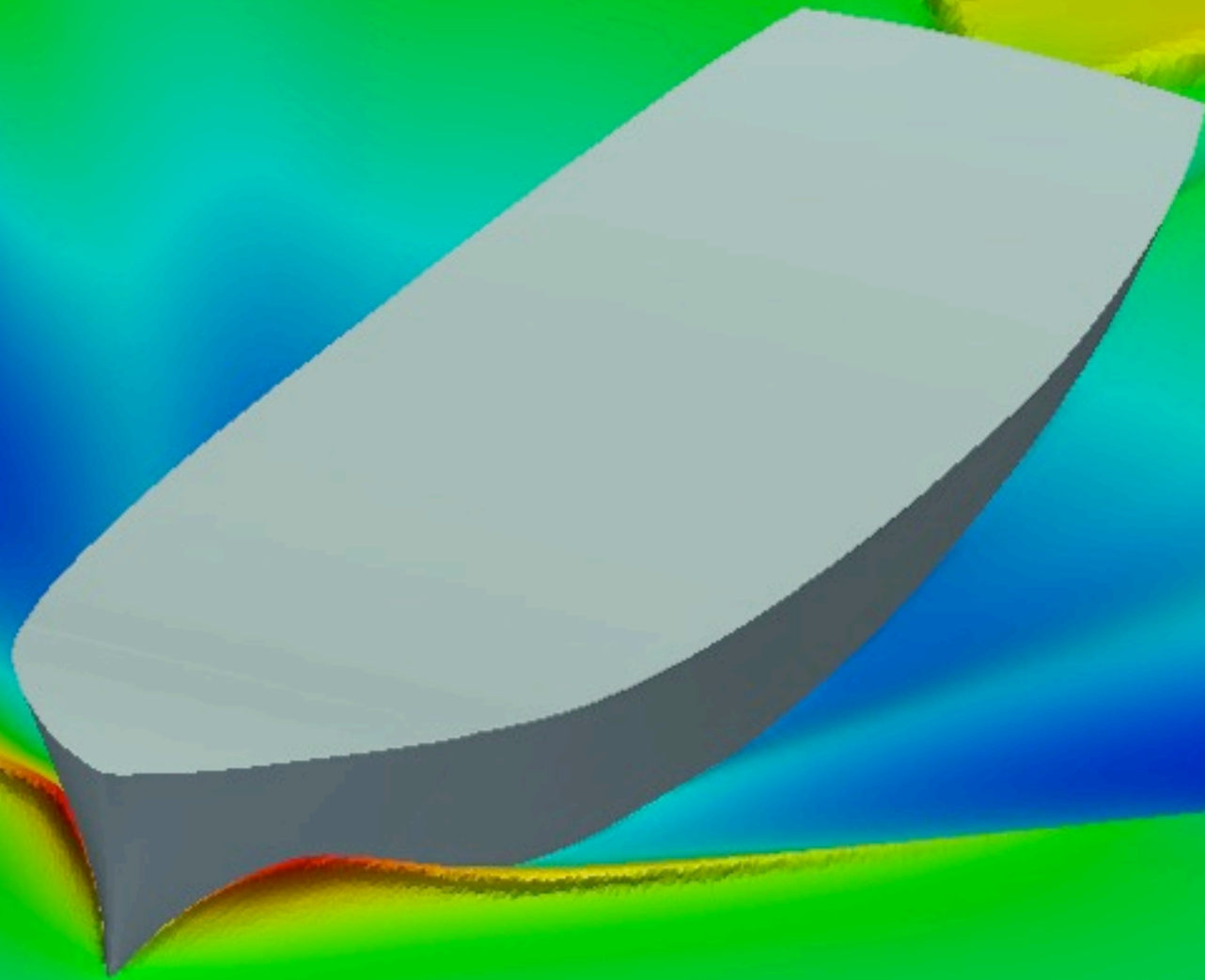
# MESH -- FR = 0.41

Icem CFD Mesh: 13.3Melem (11.8M tets + 1.5M prisms + 7K pyramids). Whole domain - No symmetry.



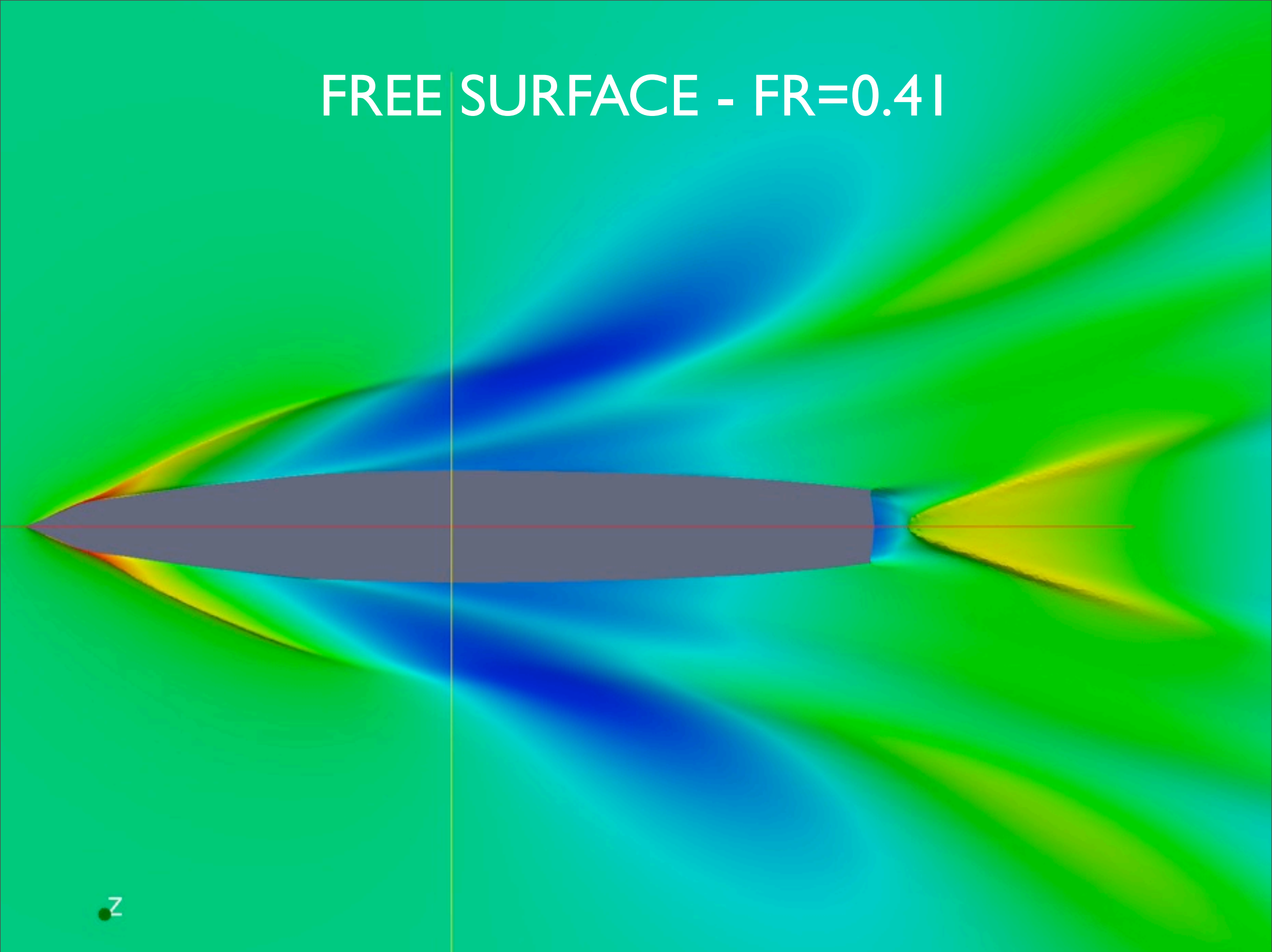


FREE SURFACE -  $FR=0.41$





# FREE SURFACE - $FR=0.41$

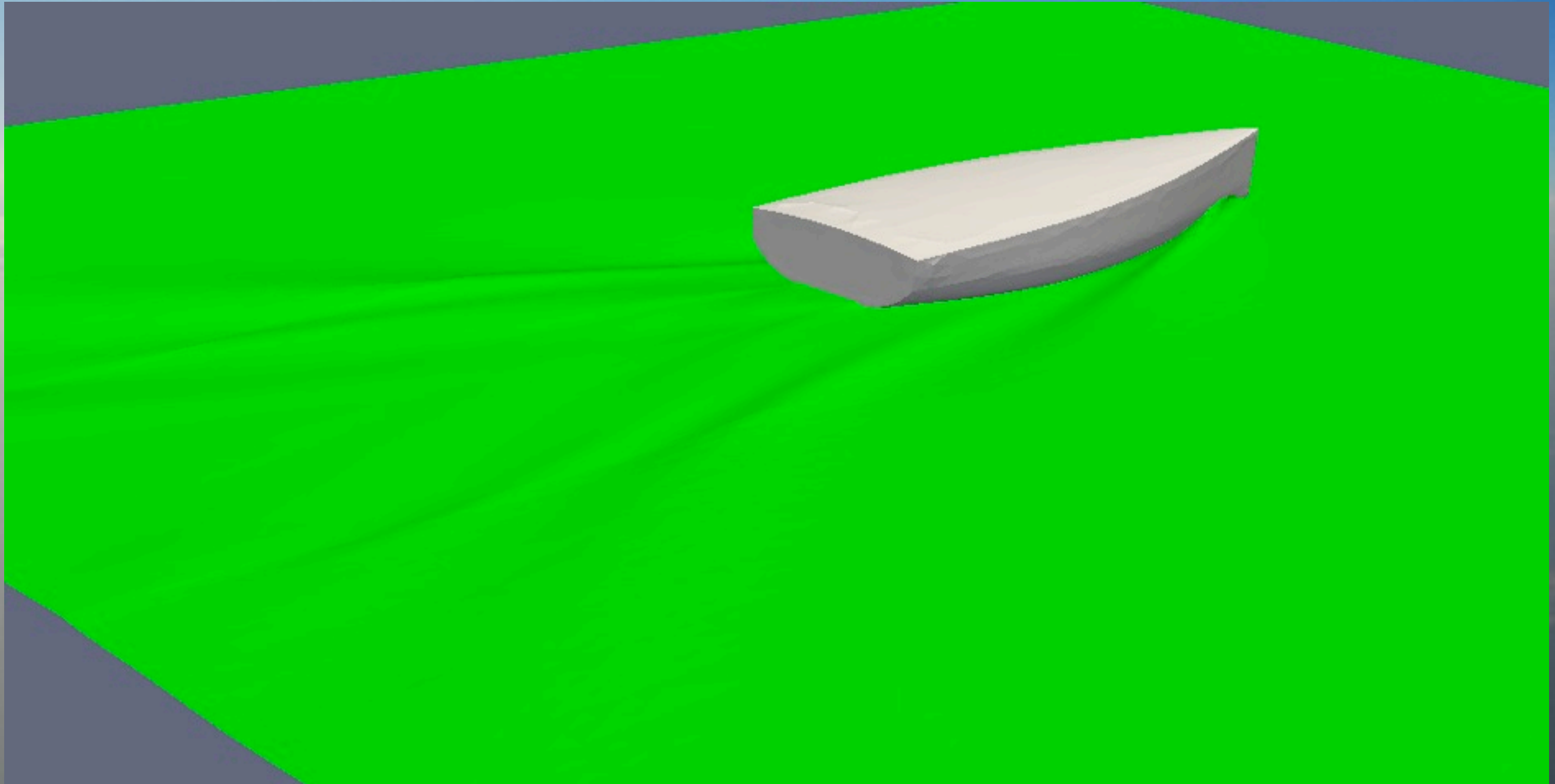


Forces on the hull  $Fr = 0.41$ 

|                        | Experimental            | Numerical |
|------------------------|-------------------------|-----------|
| $F_x[N]$ (total)       | 23.52                   | 24.54     |
| $F_{vx}[N]$ (viscous)  | 10.9 (from correlation) | 11.64     |
| $F_{px}[N]$ (pressure) |                         | 12.9      |



## Application to a race boat hull



## Application to a race boat hull

|                            |              |
|----------------------------|--------------|
| Length, $L$                | $10.85m$     |
| Velocity, $U$              | $5.144m/s$   |
| $Re = \frac{UL\rho}{\mu}$  | $4.7 * 10^7$ |
| $Fr = \frac{U}{\sqrt{gL}}$ | $0.4985$     |

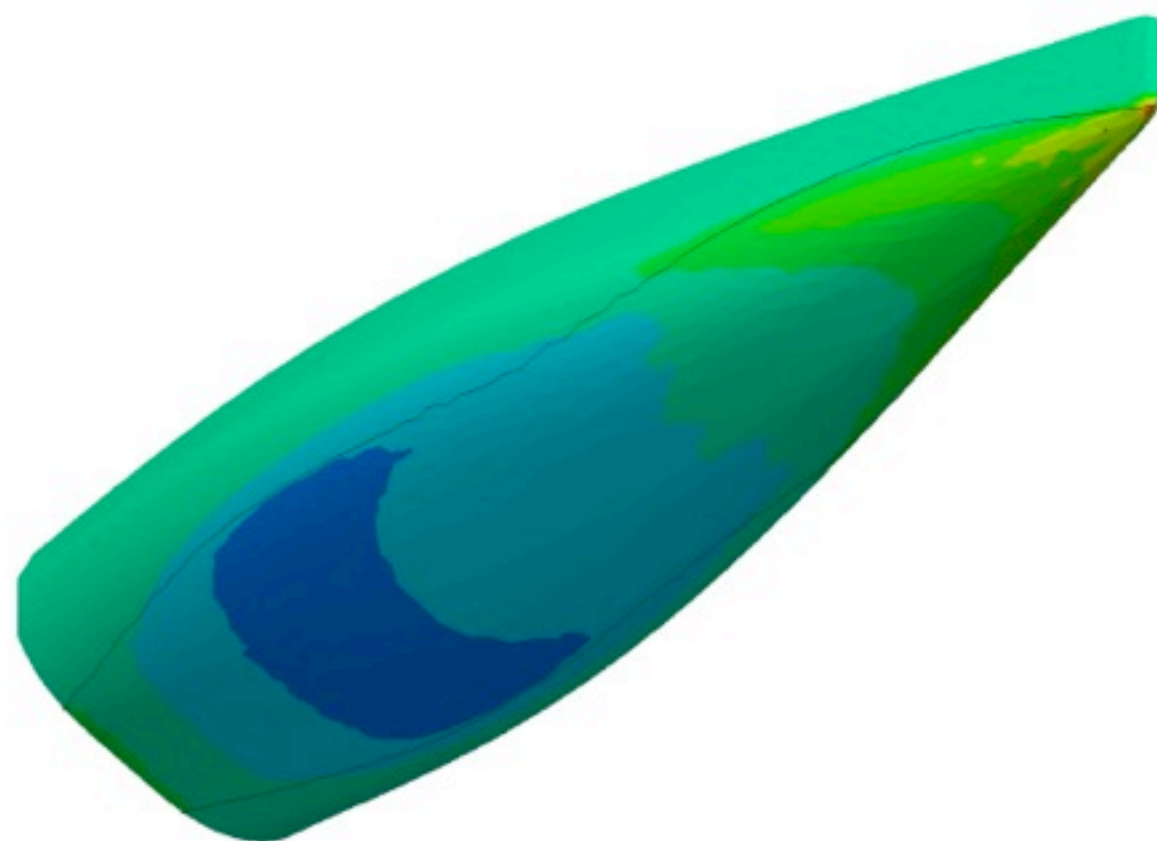
- Original mesh: 2.12 Melem (Tetras + Prisms), 0.36 Gnodes
- Divided mesh: Each element is divided so that the new element has half the size. In 3D 8 times more nodes and elements.

|                        | Original mesh | Divided mesh |
|------------------------|---------------|--------------|
| $F_x[N](total)$        | 2132.0        | 2161.6       |
| $F_{Vx}[N]$ (viscous)  | 744.0         | 748.6        |
| $F_{Px}[N]$ (pressure) | 1388.0        | 1413.0       |

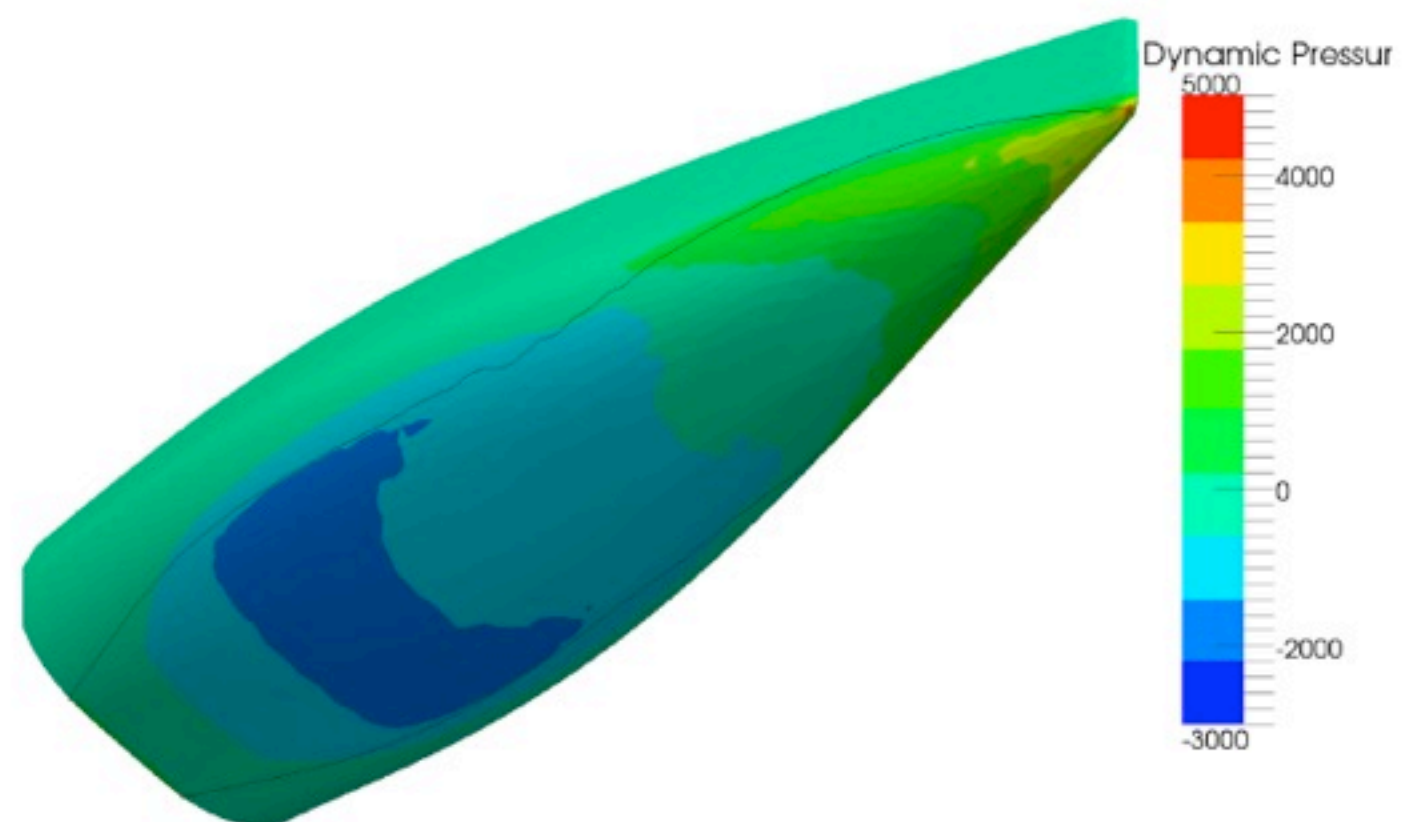


## Pressure

Original mesh

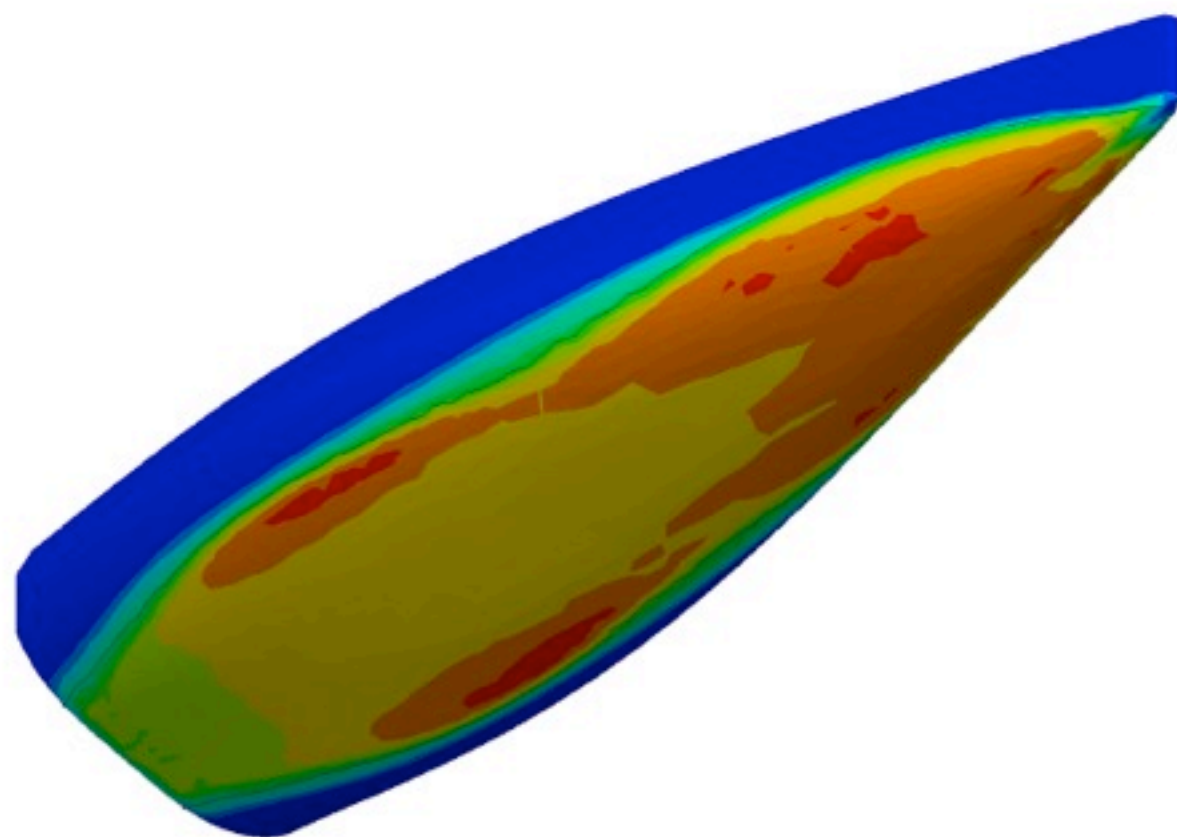


Divided mesh

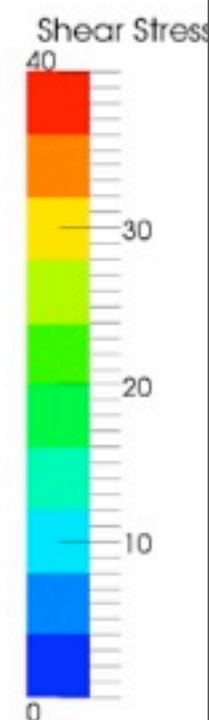
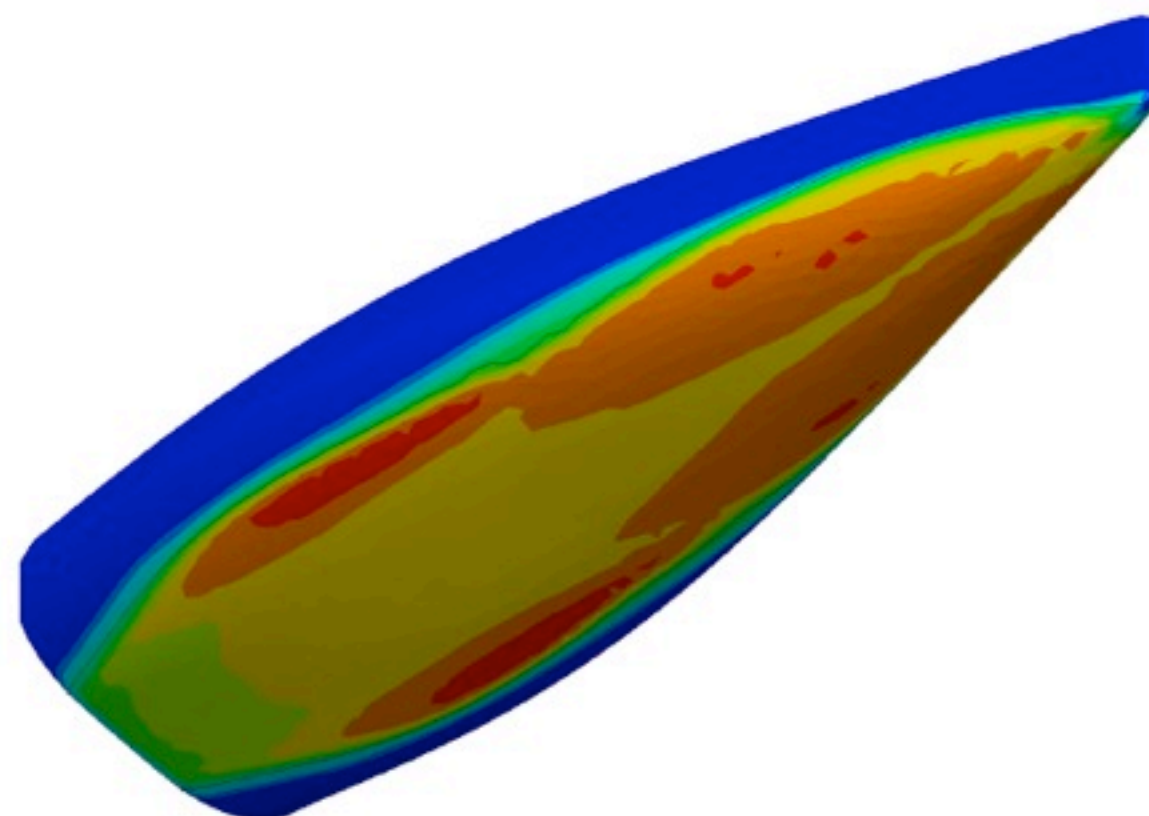


## Tangential Stress

Original mesh



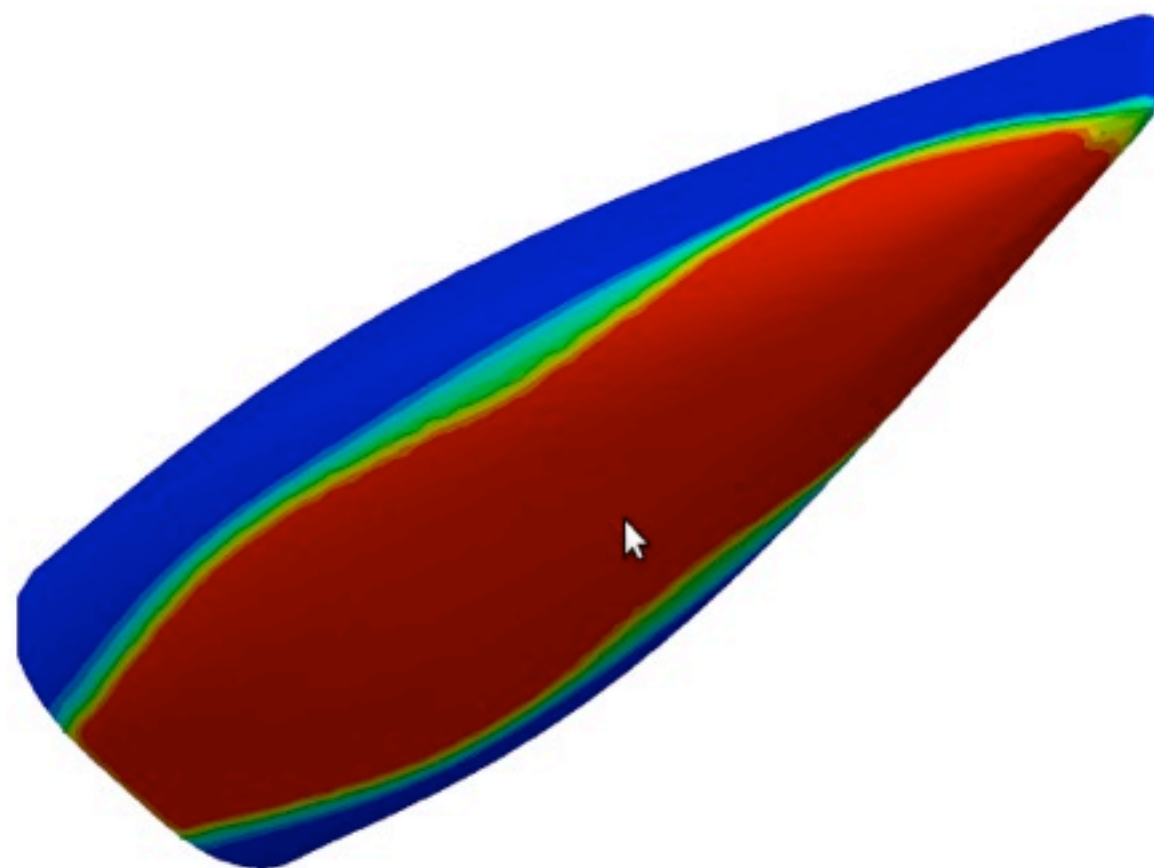
Divided mesh



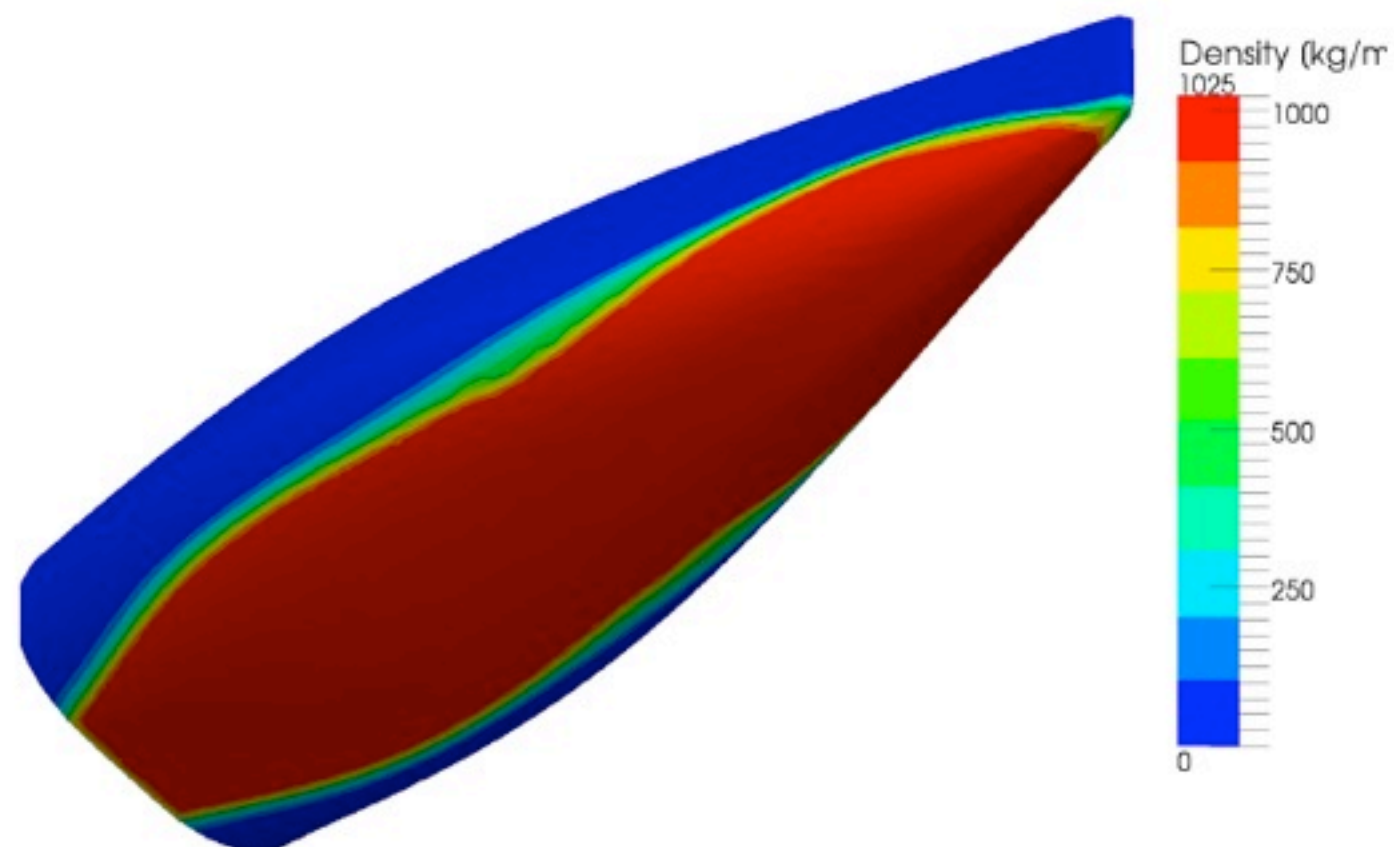


## Density

Original mesh

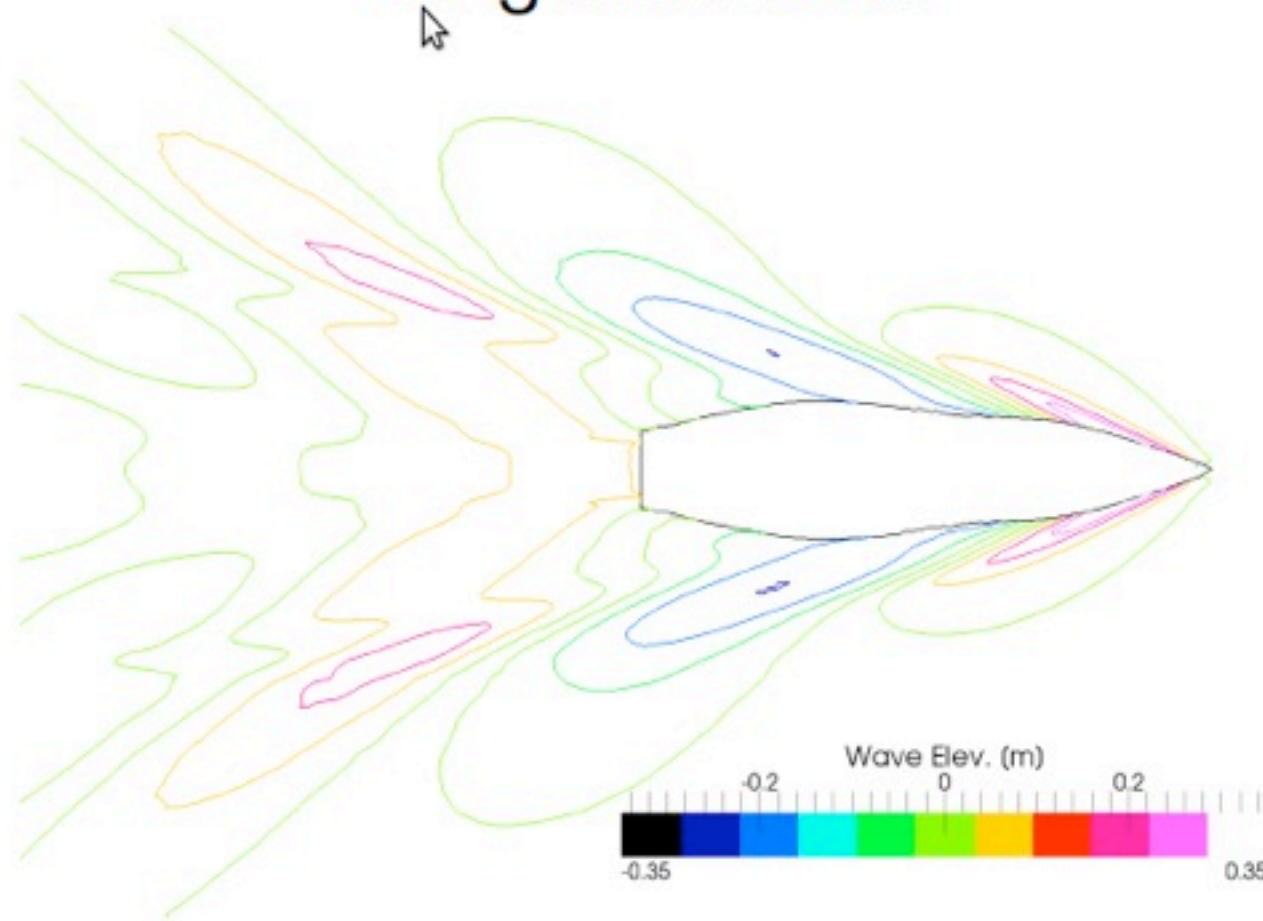


Divided mesh

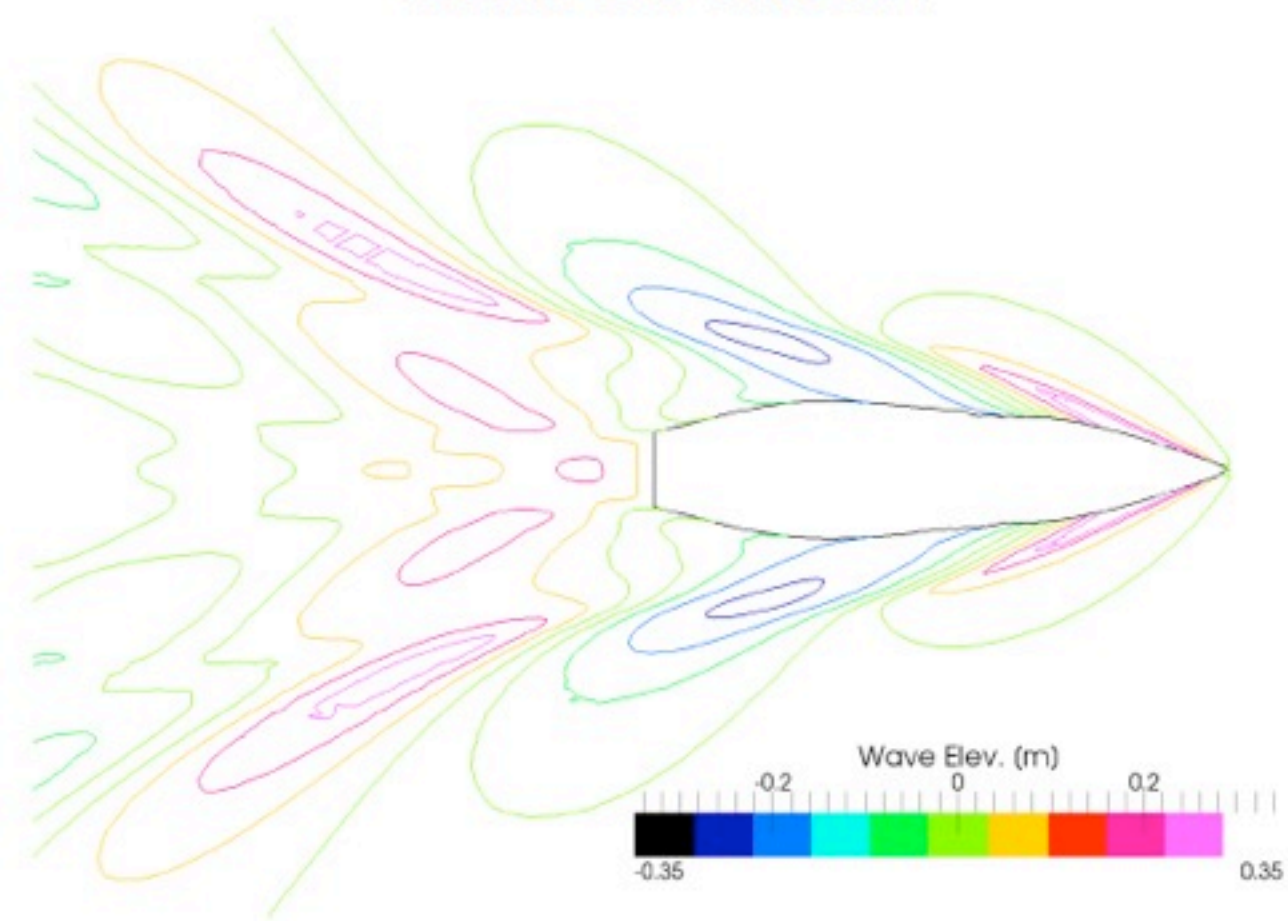


## Wave elevation

Original mesh



Divided mesh





Linear and  
angular RB  
dynamics

$$m\ddot{\mathbf{X}} = \mathbf{F},$$

$$\mathbf{R}\mathbf{I}\mathbf{R}^{-1}\dot{\boldsymbol{\Omega}} + \boldsymbol{\Omega} \times \mathbf{R}\mathbf{I}\mathbf{R}^{-1}\boldsymbol{\Omega} = \mathbf{M}_G.$$

Time  
integration  
scheme

$$\dot{\mathbf{X}}^{n+1} = \dot{\mathbf{X}}^n + \delta t \left( 1.5\ddot{\mathbf{X}}^n - 0.5\ddot{\mathbf{X}}^{n-1} \right)$$

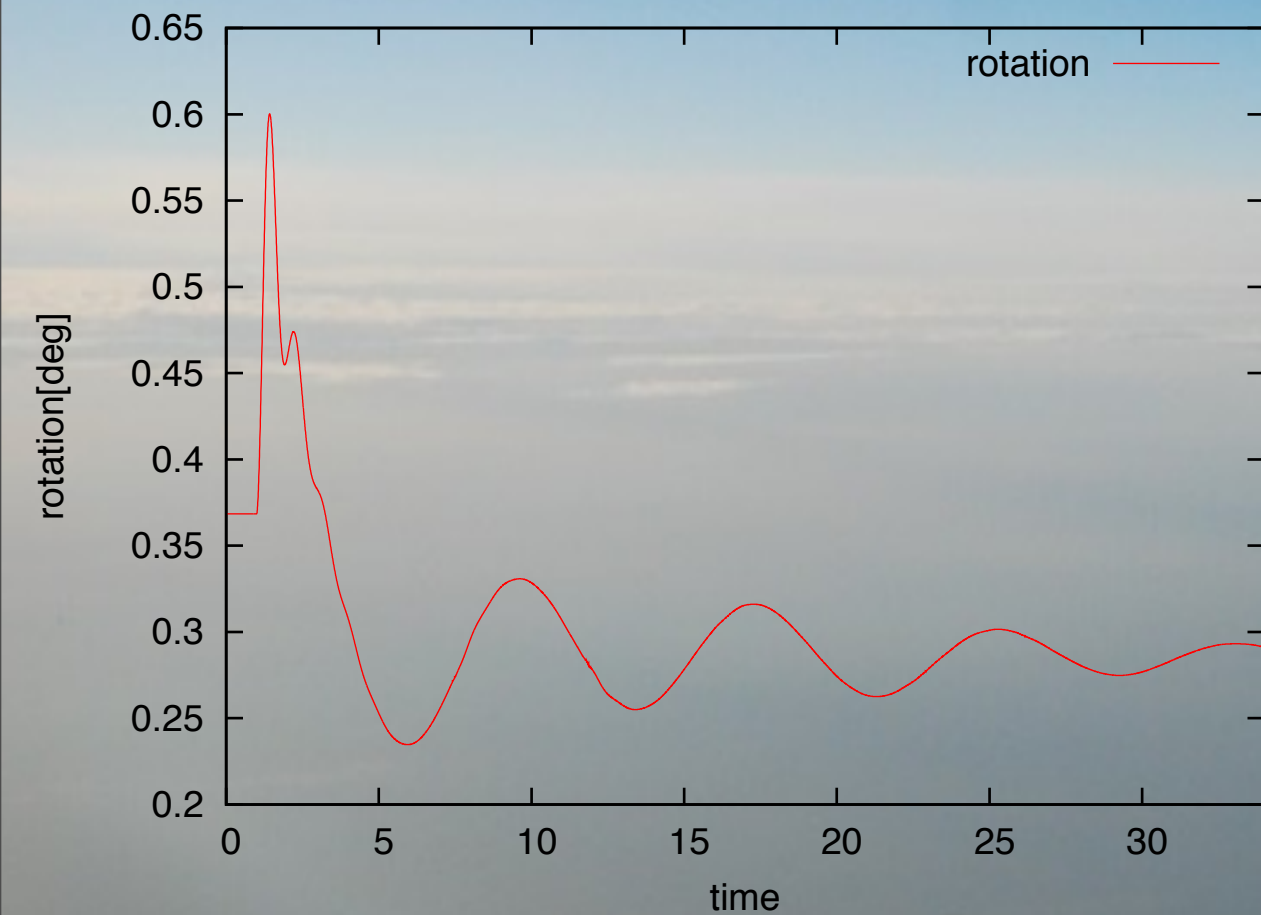
$$\mathbf{X}^{n+1} = \mathbf{X}^n + \delta t \dot{\mathbf{X}}^{n+1}.$$

## Arbitrary Lagrangian Eulerian formulation

- Convective term in NSI, TUR & LS modified accordingly.
- Rigid body nodes displaced according to RB motion.
- Weighted Laplacian for interior nodes.

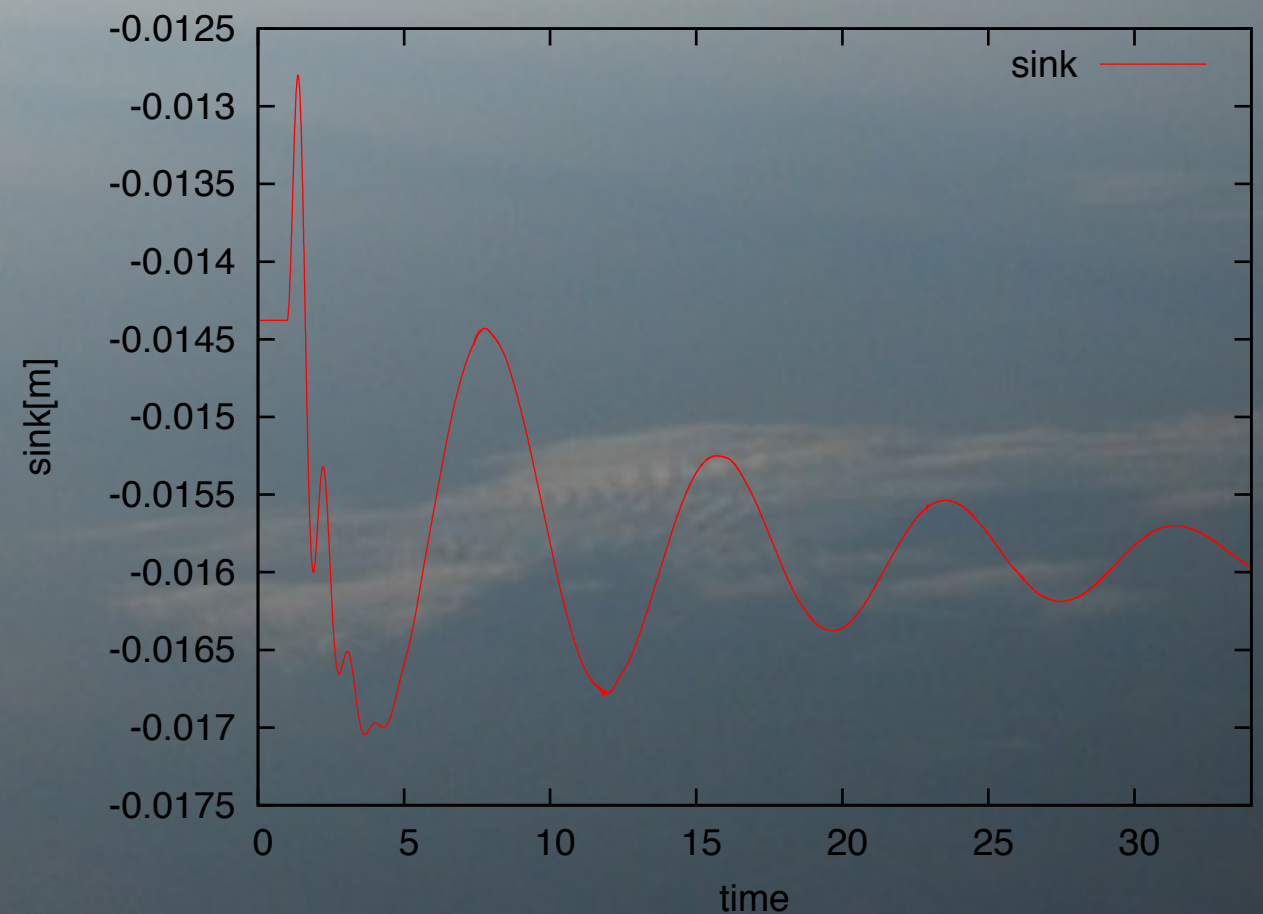


## Sink and Trim - $Fr=0.41$



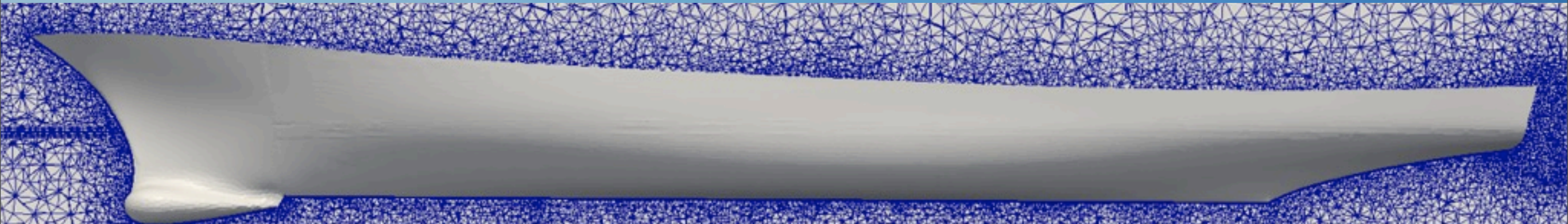
Trim (exp 0.3684)

Sink (exp 0.0143)

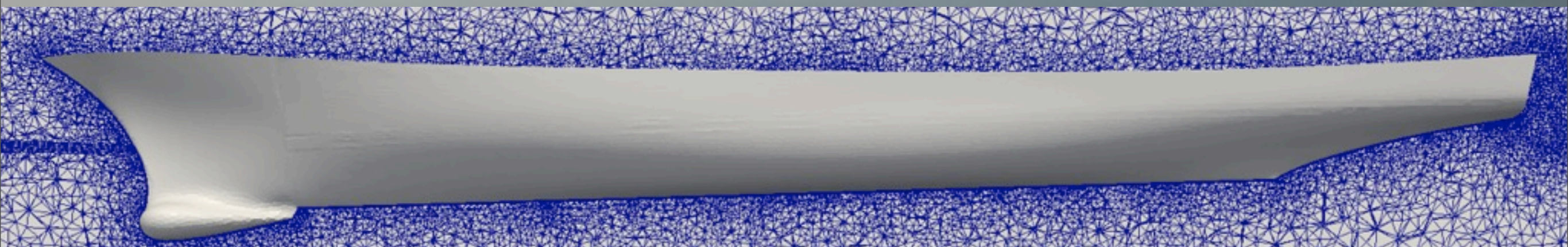




## Rigid body motion using ALE - $Fr = 0.28$

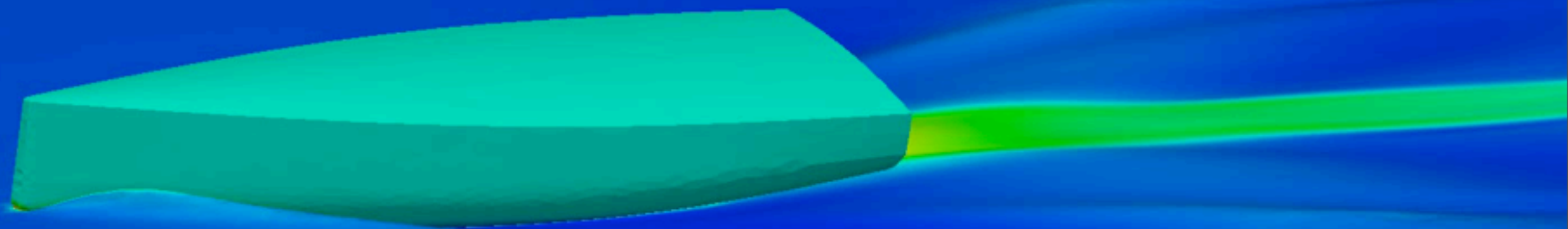


Initial mesh



Deformed mesh at  $t = 2.25s$ . Rotation magnified 3 times



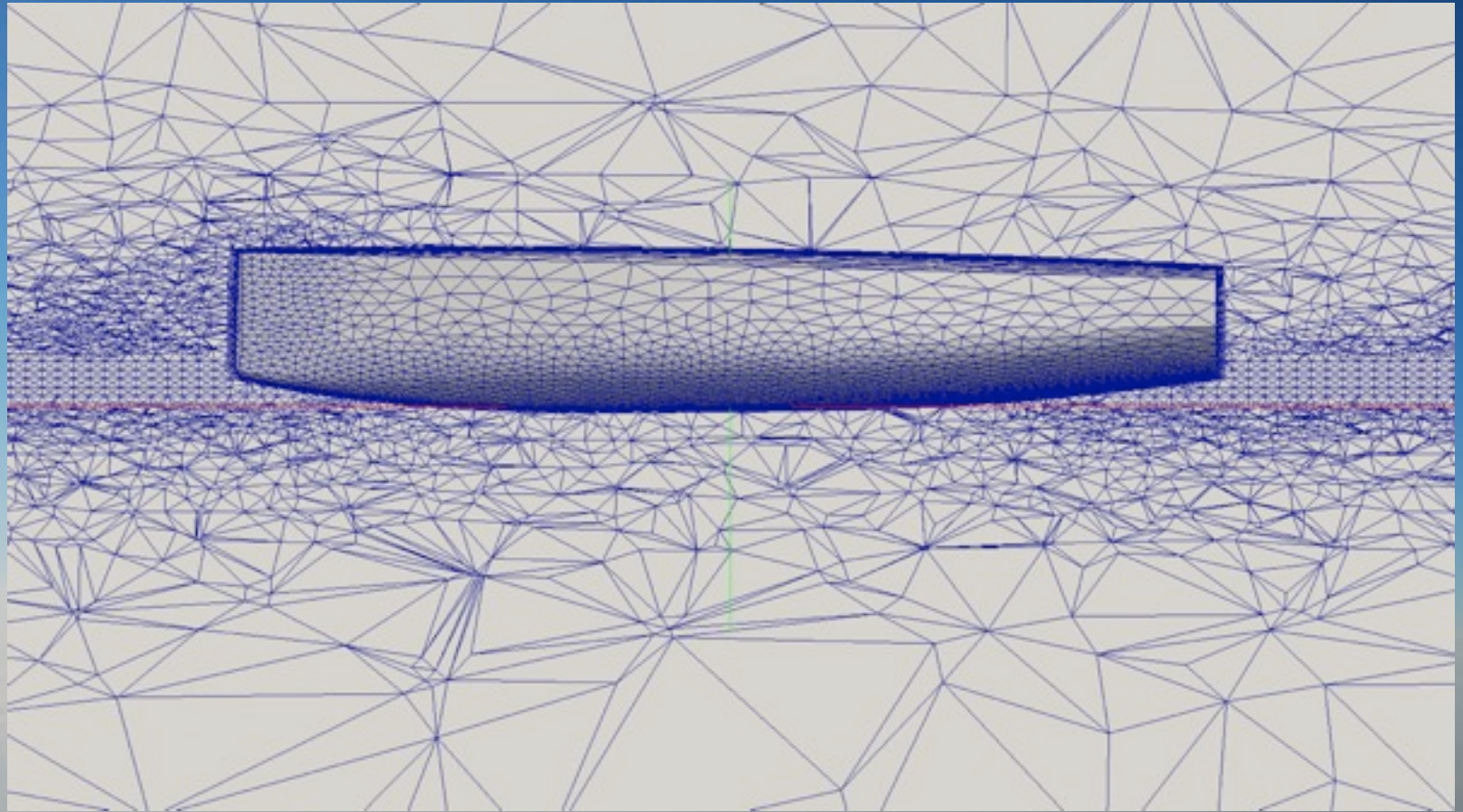


Race boat CFD

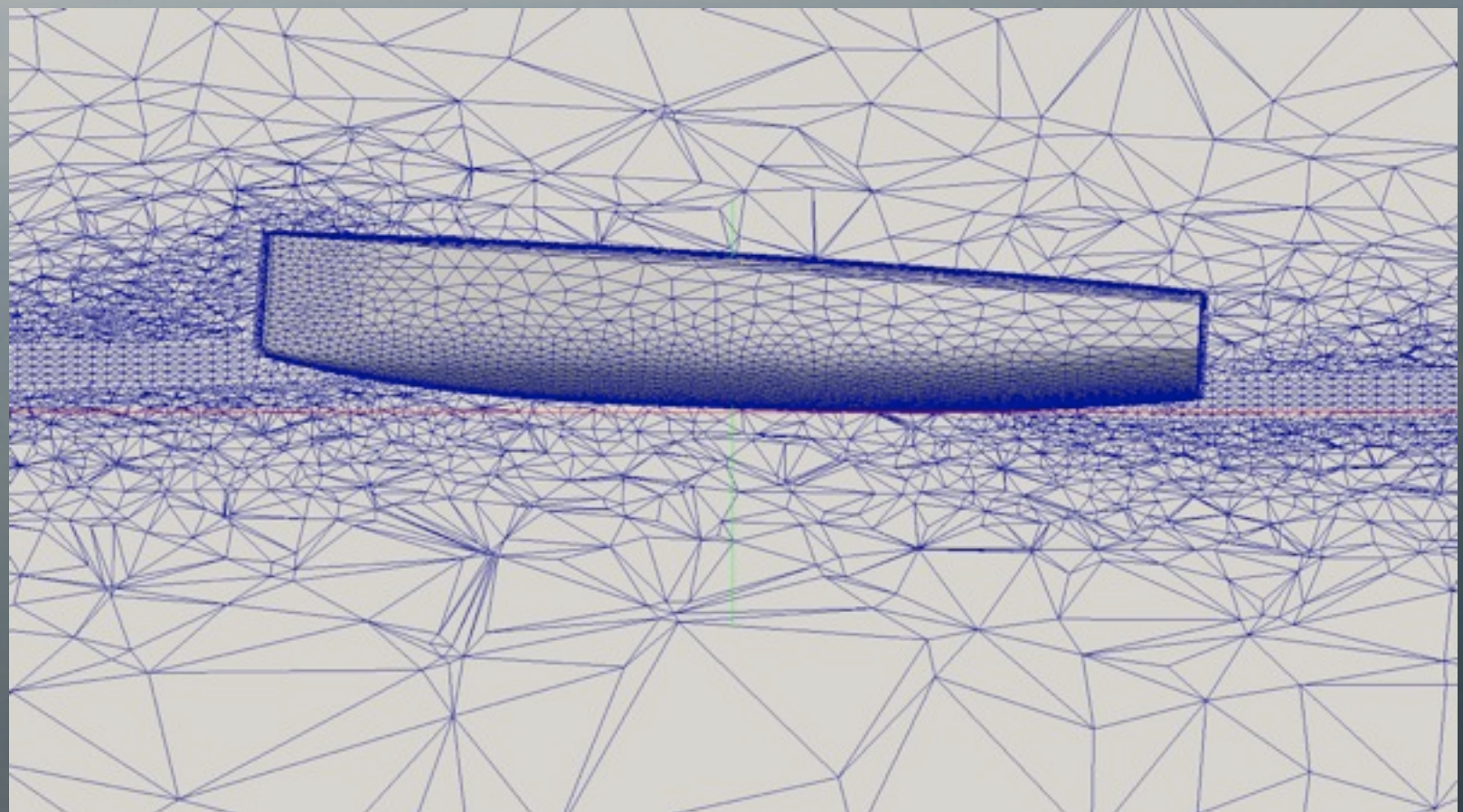


## Race boat rigid body motion using ALE

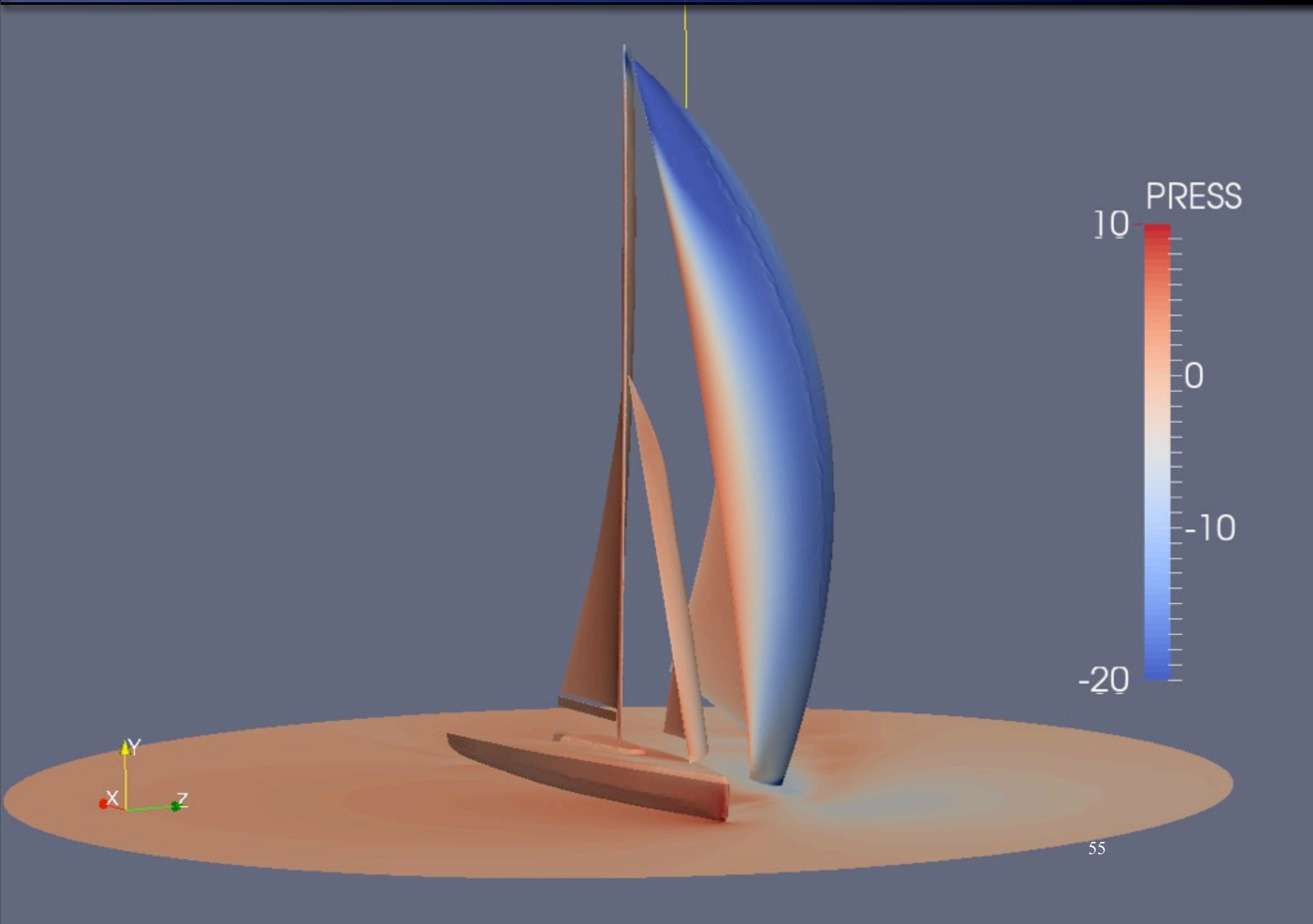
Initial mesh



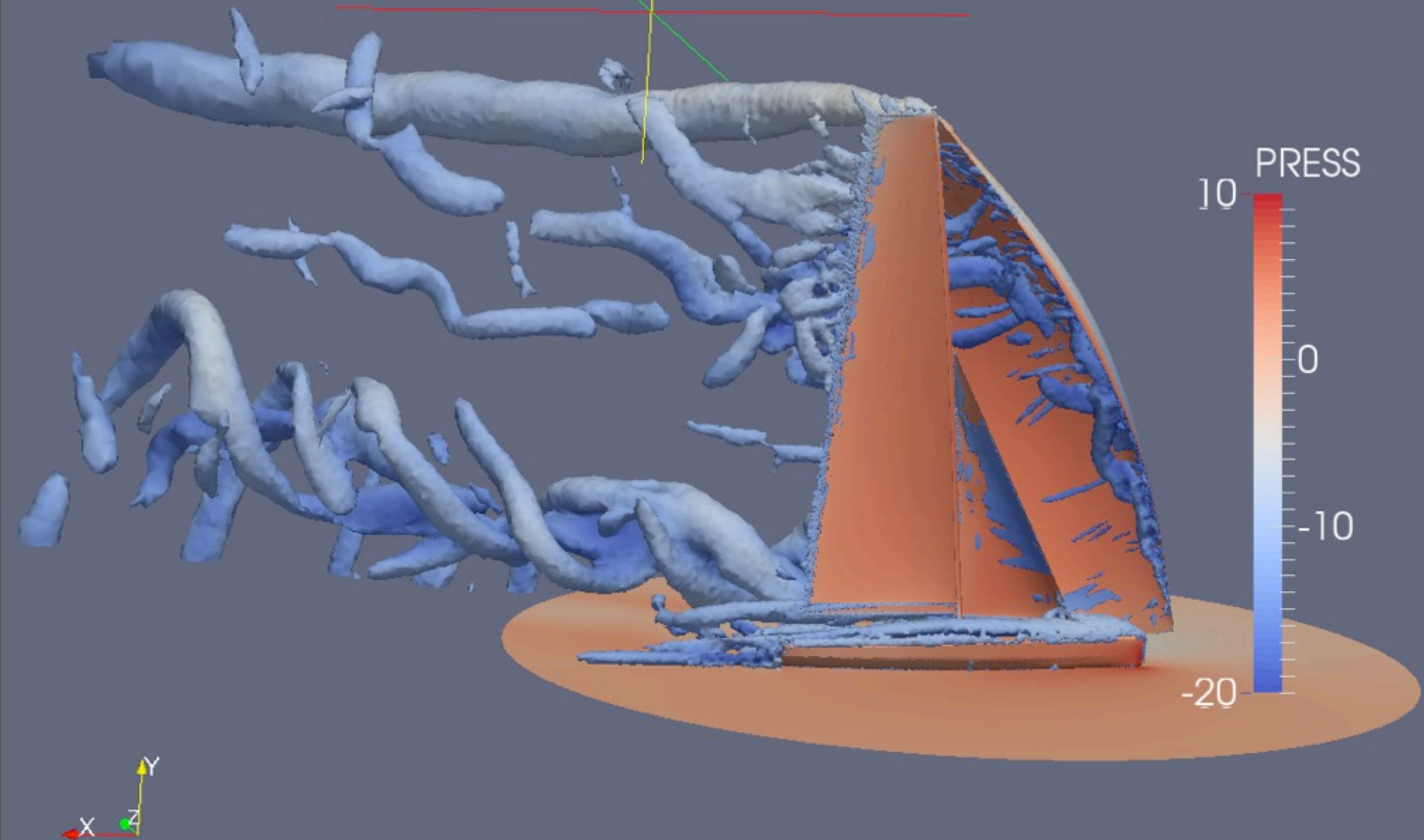
Deformed mesh at  $t = 6s$ .





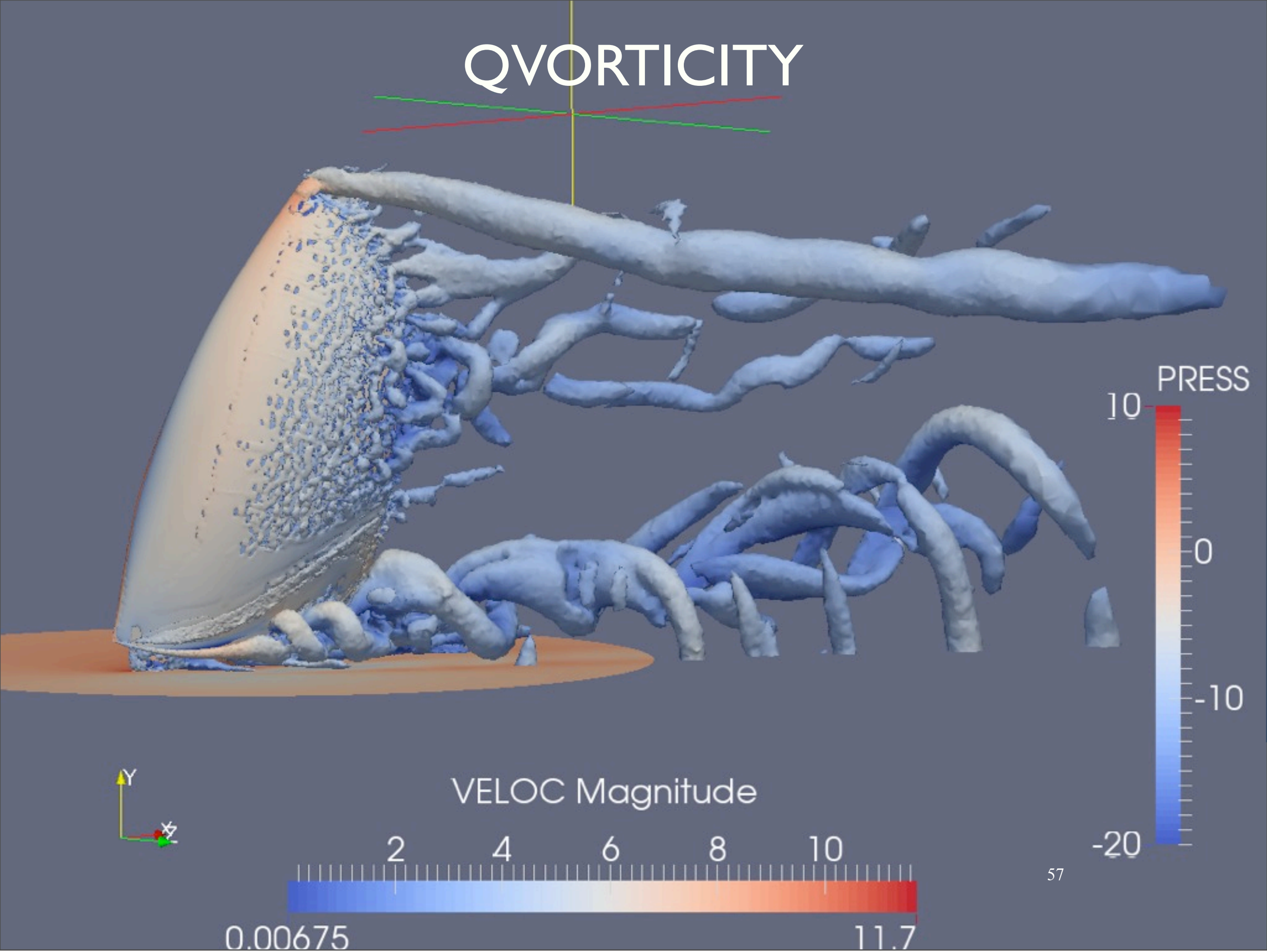


# QVORTICITY





# QVORTICITY



An aerial photograph showing a vast, flat landscape, likely a coastal plain or a large body of water, under a clear blue sky. The horizon is visible in the distance, and the overall scene is serene and expansive.

Thanks!!!